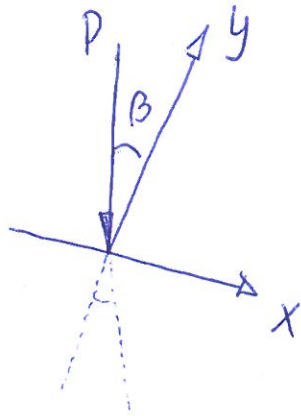


2.1



$$P = 800 \text{ N}$$

$$\tan \beta = \frac{3}{4}$$

$$P_x = P \cdot \sin \beta$$

$$P_y = -P \cdot \cos \beta$$

$$\beta = 36,86^\circ$$

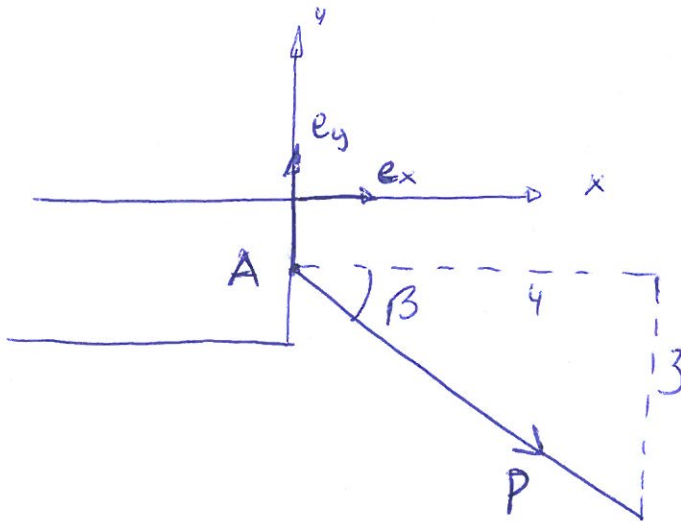
$$P_x = 800 \cdot \sin 36,86 = 480 \text{ N}$$

$$P_y = -800 \cdot \cos \beta = -640 \text{ N}$$

Svar : $P_x = 480 \text{ N}$, $P_y = -640 \text{ N}$

$$\vec{P} = (480 \hat{e}_x - 640 \hat{e}_y) \text{ N}$$

2.2



$$P = 10 \text{ kN}$$

Basvektorer \vec{e}_x \vec{e}_y

$$\tan \beta = \frac{3}{4}$$

$$\beta = 36,86^\circ$$

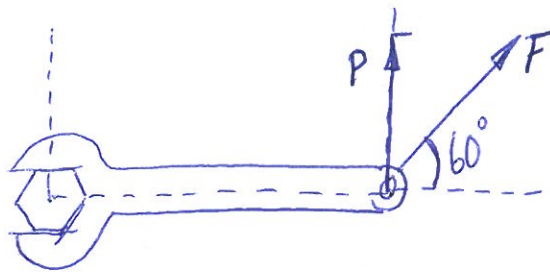
$$\sin \beta = \frac{3}{5}$$

$$\cos \beta = \frac{4}{5}$$

$$\vec{P} = (P \cdot \cos \beta \cdot \vec{e}_x - P \sin \beta \cdot \vec{e}_y) = (10 \cdot \cos \beta \vec{e}_x - 10 \sin \beta \vec{e}_y)$$

$$\vec{P} = (8 \vec{e}_x - 6 \vec{e}_y) \text{ kN}$$

2,3



$$F_x = ? \quad F = ?$$

$$F_y = P$$

$$F_x = F \cdot \cos 60$$

$$F_x = \frac{F}{2}$$

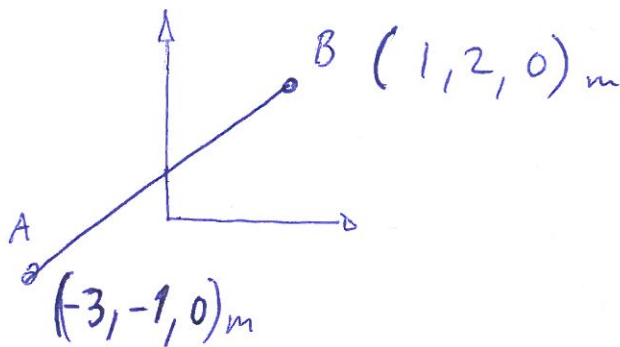
$$F_y = F \cdot \sin 60$$

$$P = \frac{\sqrt{3}}{2} F \Rightarrow F = \frac{2P}{\sqrt{3}}$$

$$F_x = \frac{2P}{\sqrt{3}} \cdot \frac{1}{2} = \frac{P}{\sqrt{3}}$$

Svar: $F_x = \frac{P}{\sqrt{3}} \quad F = \frac{2P}{\sqrt{3}}$

2,4



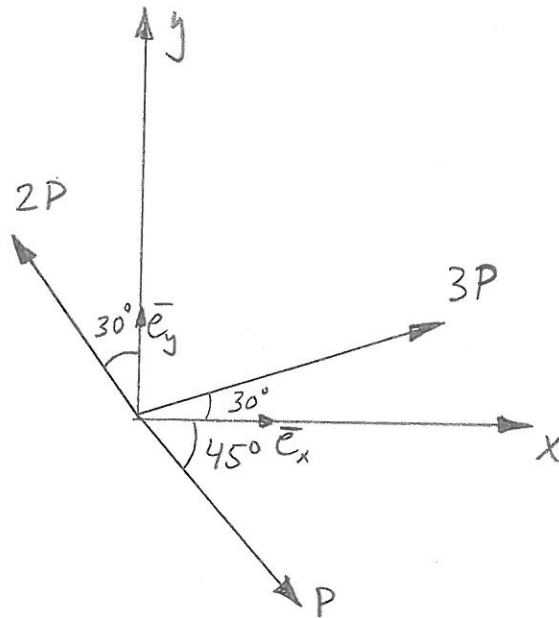
$$\bar{e}_{AB} = \frac{\bar{r}_B - \bar{r}_A}{|\bar{r}_B - \bar{r}_A|} = \frac{4, 3, 0}{\sqrt{4^2 + 3^2}} = \frac{(4, 3, 0)}{5}$$

$$\bar{e}_{AB} = \frac{1}{5} (4, 3, 0)$$

$$S_A = 50 \cdot \left(\frac{4}{5} \bar{e}_x + \frac{3}{5} \bar{e}_y \right) = (40 \bar{e}_x + 30 \bar{e}_y)$$

$$S_B = (-40 \bar{e}_x + 30 \bar{e}_y) \text{ N}$$

2.5



$$\vec{F}_1 = P \cos 45^\circ \vec{e}_x - P \sin 45^\circ \vec{e}_y$$

$$\vec{F}_2 = 3P \cos 30^\circ \vec{e}_x + 3P \sin 30^\circ \vec{e}_y$$

$$\vec{F}_3 = -2P \sin 30^\circ \vec{e}_x + 2P \cos 30^\circ \vec{e}_y$$

Kan också skrivas som.

$$\vec{F}_1 = P \cdot \frac{\sqrt{2}}{2} \vec{e}_x - P \cdot \frac{\sqrt{2}}{2} \vec{e}_y$$

$$\vec{F}_2 = 3P \cdot \frac{\sqrt{3}}{2} \vec{e}_x + 3P \frac{1}{2} \vec{e}_y$$

$$\vec{F}_3 = -2P \frac{1}{2} \vec{e}_x + 2 \frac{\sqrt{3}}{2} \vec{e}_y$$

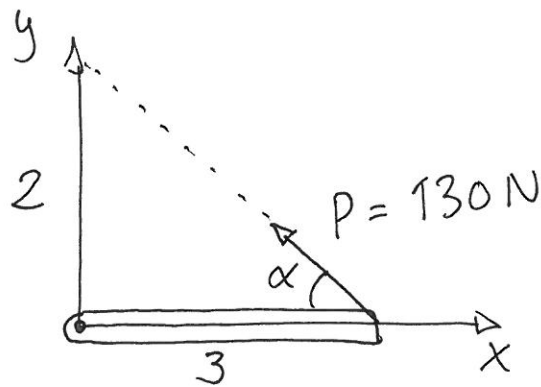
eller
$$\vec{F}_1 = \frac{\sqrt{2}P}{2} (\vec{e}_x - \vec{e}_y)$$

$$\vec{F}_2 = \frac{3}{2}P (\sqrt{3} \vec{e}_x + \vec{e}_y)$$

$$\vec{F}_3 = 2P (-\vec{e}_x + \sqrt{3} \vec{e}_y)$$

$$\vec{F} = \vec{F}_1 + \vec{F}_2 + \vec{F}_3 = \left(\frac{\sqrt{2}P}{2} + \frac{3\sqrt{3}P}{2} - P \right) \vec{e}_x + \left(-\frac{\sqrt{2}P}{2} + \frac{3P}{2} + \sqrt{3}P \right) \vec{e}_y$$

2.6)



$$\tan \alpha = \frac{2}{3}$$

$$\sin \alpha = \frac{2}{\sqrt{13}}$$

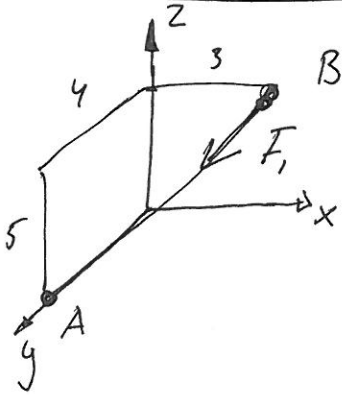
$$\cos \alpha = \frac{3}{\sqrt{13}}$$

$$\alpha = 33,69^\circ$$

$$x: \rightarrow : -130 \cdot \cos 33,69 = -108,16 \text{ N}$$

$$y: \uparrow : 130 \cdot \sin 33,69 = 72,1 \text{ N}$$

2.10)



$$F_1 = 300 \text{ N}$$

$$\vec{F}_1 = F_1 \cdot \vec{e}_{BA}$$

$$\vec{e}_{BA} = \frac{(4, 0, 0) - (0, 3, 5)}{|\vec{r}_A - \vec{r}_B|}$$

$$\vec{e}_{BA} = \frac{(4, -3, -5)}{\sqrt{4^2 + 3^2 + (-5)^2}} = \frac{(4, -3, -5)}{\sqrt{50}}$$

$$\vec{e}_{BA} = \frac{(4, -3, -5)}{\sqrt{25} \cdot \sqrt{2}} = \frac{(4, -3, -5)}{5 \cdot \sqrt{2}}$$

$$\vec{F}_1 = \frac{300}{5 \cdot \sqrt{2}} (4, -3, -5) \text{ N}$$

$$\vec{F}_1 = \frac{60}{\sqrt{2}} (4, -3, -5) \text{ N}$$

Repets kraft på mannen
är $-\vec{F}_1$

$$2.11 \quad \bar{e}_{AB} = \frac{(0,5,2) - (3,4,0)}{\sqrt{14}} = \frac{-3,1,2}{\sqrt{14}}$$

$$\bar{e}_{AC} = \frac{(4,0,5) - (3,4,0)}{\sqrt{42}} = \frac{(1,-4,5)}{\sqrt{42}}$$

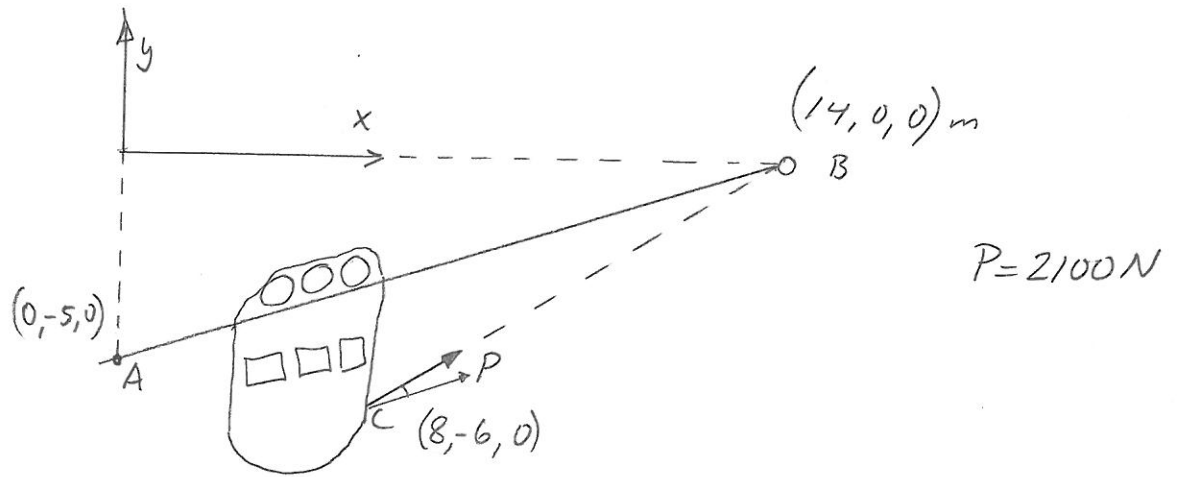
$$\bar{e}_{AB} \cdot \bar{e}_{AC} = \cos \beta$$

$$\cos \beta = \frac{1}{\sqrt{42}} \cdot \frac{-3}{\sqrt{14}} + \frac{1(-4)}{\sqrt{42} \cdot \sqrt{14}} + \frac{10}{\sqrt{42} \cdot \sqrt{14}}$$

$$\cos \beta = \frac{-3}{\sqrt{14} \cdot \sqrt{3} \cdot \sqrt{14}} + \frac{4}{\sqrt{3} \cdot \sqrt{14} \cdot \sqrt{14}} + \frac{10}{\sqrt{14} \cdot \sqrt{14} \cdot \sqrt{3}}$$

$$\cos \beta = \frac{3}{14 \cdot \sqrt{3}}$$

2,12



$$\bar{e}_{CB} = \frac{(14, 0, 0) - (8, -6, 0)}{\sqrt{6^2 + 6^2}} = \frac{(6, 6, 0)}{\sqrt{72}}$$

$$\bar{e}_{AB} = \frac{(14, 0, 0) - (0, -5, 0)}{\sqrt{14^2 + 5^2}} = \frac{(14, 5, 0)}{\sqrt{221}}$$

$$\bar{e}_{CB} = \frac{(6, 6, 0)}{\sqrt{72}} = \frac{(6, 6, 0)}{\sqrt{36 \cdot 2}} = \frac{6}{6 \cdot \sqrt{2}} (1, 1, 0) = \frac{1}{\sqrt{2}} (1, 1, 0)$$

$$\bar{P} = P \cdot \bar{e}_{CB} = \frac{2100}{\sqrt{2}} (1, 1, 0)$$

Kraftens komponent avseende på linan AB

$$P_{\text{komp}} = \bar{P} \cdot \bar{e}_{AB} = P \cdot \bar{e}_{CB} \cdot \bar{e}_{AB} = \frac{2100}{\sqrt{2} \cdot \sqrt{221}} (14 + 5 + 0) \text{ N}$$

$$= \frac{19}{\sqrt{442}} 2100 \text{ N}$$

2.13

$$F_1 = 340 \text{ N}$$

$$F_2 = 420 \text{ N}$$

$$F_3 = 390 \text{ N}$$

Ta reda på kraftvektorerna \vec{F}_1 , \vec{F}_2 och \vec{F}_3

först riktningsvektorerna. \hat{e}_{F_1} \hat{e}_{F_2} \hat{e}_{F_3}

$$\hat{e}_{F_1} \quad \hat{e}_{F_2} \quad \hat{e}_{F_3}$$

$$\hat{e}_{F_1} = \frac{(9, -8, 0) - (0, 0, 12)}{|(9, -8, 0) - (0, 0, 12)|} = \frac{(9, -8, 12)}{\sqrt{81+64+144}} = \frac{(9, -8, 12)}{17}$$

$$\Rightarrow \vec{F}_{F_1} = \hat{e}_{F_1} \cdot F_1 = \frac{(9, -8, 12)}{17} \cdot 340 = 20(9, -8, 12) \text{ N}$$

$$\hat{e}_{F_2} = \frac{(4, 6, 0) - (0, 0, 12)}{\sqrt{16+36+144}} = \frac{(4, 6, -12)}{14}$$

$$\hat{e}_{F_3} = \frac{(-5, 0, 0) - (0, 0, 12)}{\sqrt{25+144}} = \frac{(-5, 0, -12)}{13}$$

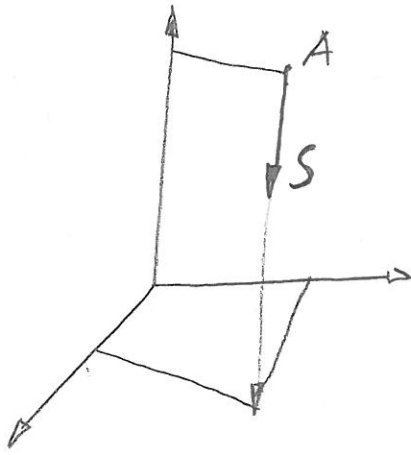
$$\vec{F}_{F_2} = \hat{e}_{F_2} \cdot F_2 = \frac{(4, 6, -12)}{14} \cdot 420 = 30(4, 6, -12) \text{ N}$$

$$\vec{F}_{F_3} = \hat{e}_{F_3} \cdot F_3 = \frac{(-5, 0, -12)}{13} \cdot 390 = (150, 0, -360)$$

$$\vec{R} = (\Sigma F_x, \Sigma F_y, \Sigma F_z) = (180+120-150, -160+180, -240-360-360)$$

$$\vec{R} = (150, 20, -960) \text{ N}$$

2.14)



$$\vec{S} = ?$$

$$S = 0,9 \text{ kN}$$

$$\vec{r}_A = (0, 2, 7)$$

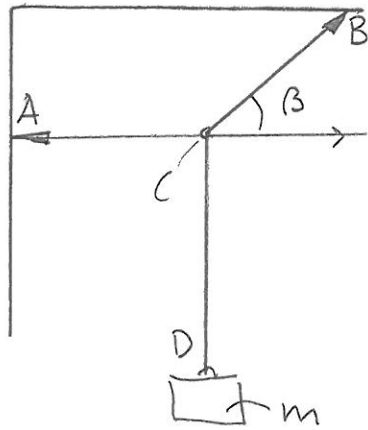
$$\vec{r}_B = (4, 6, 0)$$

$$\vec{S} = S \cdot \hat{e}_{AB}$$

$$\begin{aligned} \hat{e}_{AB} &= \frac{\vec{r}_B - \vec{r}_A}{|\vec{r}_B - \vec{r}_A|} = \frac{(4, 6, 0) - (0, 2, 7)}{|(4, 6, 0) - (0, 2, 7)|} \\ &= \frac{(4, 4, -7)}{\sqrt{4^2 + 4^2 + 7^2}} = \frac{1}{9} (4, 4, -7) \end{aligned}$$

$$\vec{S} = \frac{0,9}{9} * (4, 4, -7) \text{ kN} = (0,4, 0,4, -0,7) \text{ kN}$$

2.17


 S_{CB}
 $A S_{CA}$

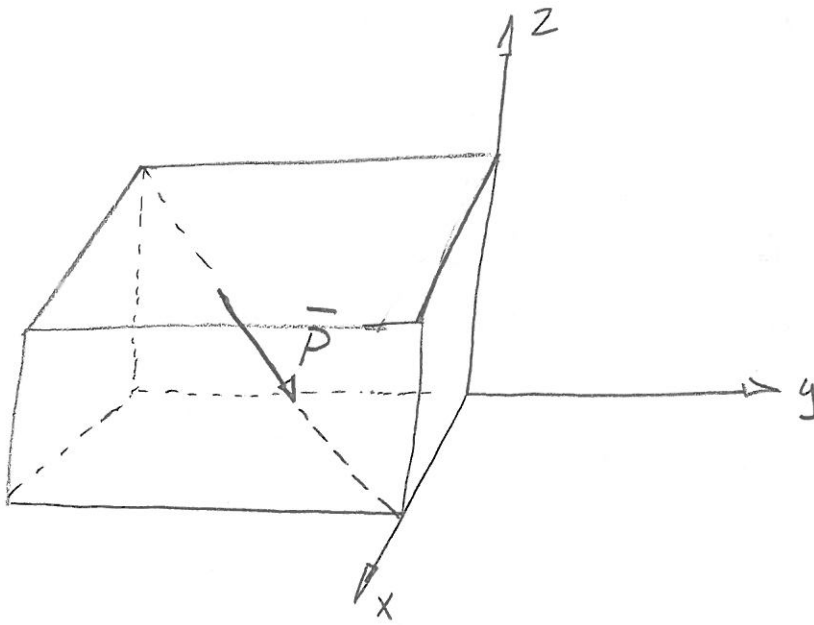
$$\rightarrow \Delta : S_{CB} \cos \beta - S_{CA} = 0$$

$$\uparrow \Delta : S_{CB} \sin \beta - mg = 0$$

$$S_{CB} = \frac{mg}{\sin \beta}$$

$$S_{CA} = \frac{mg \cos \beta}{\sin \beta} = \frac{mg}{\tan \beta}$$

2,19)



$$P = 200 \text{ N}$$

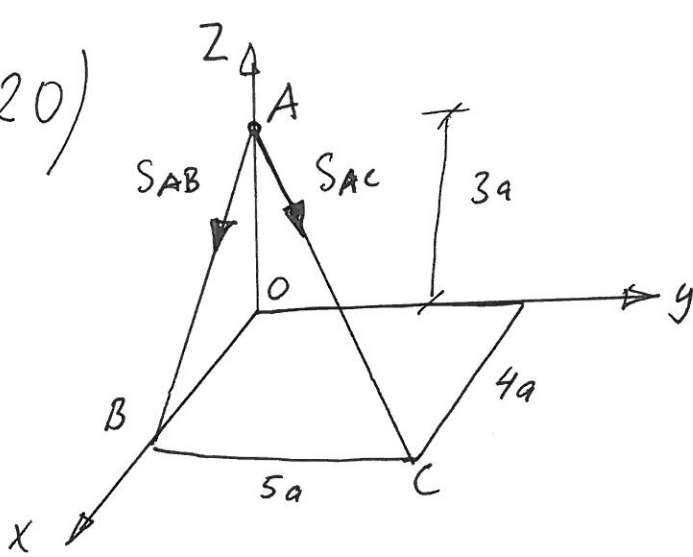
$$\vec{F}_{AB} = (2, 0, 0) - (0, -4, 3) = (2, 4, -3)$$

$$\hat{e}_{AB} = \frac{(2, 4, -3)}{\sqrt{2^2 + 4^2 + 3^2}} = \frac{1}{\sqrt{29}} (2, 4, -3)$$

$$\vec{P} = P \cdot \hat{e}_{AB} = \frac{200}{\sqrt{29}} (2, 4, -3) \text{ N}$$

$$\vec{P} = \frac{200}{\sqrt{29}} (2\hat{e}_x + 4\hat{e}_y - 3\hat{e}_z) \text{ N}$$

2.20)

Sökt total kraften
vid A = R.

$$S_{AB} = P$$

$$S_{AC} = \sqrt{2}P$$

$$\vec{S}_{AB} = S_{AB} \cdot \hat{e}_{AB}$$

$$\vec{S}_{AC} = S_{AC} \cdot \hat{e}_{AC}$$

$$\hat{e}_{AB} = \frac{(\vec{r}_B - \vec{r}_A)}{|\vec{r}_B - \vec{r}_A|} = \frac{(4a, 0, 0) - (0, 0, 3a)}{|(4a, 0, 0) - (0, 0, 3a)|} = \frac{(4a, 0, -3a)}{\sqrt{(4a)^2 + (3a)^2}}$$

$$\hat{e}_{AB} = \frac{(4a, 0, -3a)}{\sqrt{16a^2 + 9a^2}} = \frac{(4a, 0, -3a)}{5a} = \left(\frac{4}{5}, 0, -\frac{3}{5}\right)$$

$$\hat{e}_{AC} = \frac{\vec{r}_C - \vec{r}_A}{|\vec{r}_C - \vec{r}_A|} = \frac{(4a, 5a, 0) - (0, 0, 3a)}{|(4a, 5a, 0) - (0, 0, 3a)|} = \frac{(4a, 5a, -3a)}{\sqrt{16a^2 + 25a^2 + 9a^2}}$$

$$\hat{e}_{AC} = \left(\frac{4}{5\sqrt{2}}, \frac{1}{\sqrt{2}}, -\frac{3}{5\sqrt{2}}\right)$$

$$\vec{S}_{AB} = P \cdot \left(\frac{4}{5}, 0, -\frac{3}{5}\right) = \left(\frac{4P}{5}, 0, -\frac{3P}{5}\right)$$

$$\vec{S}_{AC} = \sqrt{2}P \left(\frac{4}{5\sqrt{2}}, \frac{1}{\sqrt{2}}, -\frac{3}{5\sqrt{2}}\right) = \left(\frac{4P}{5}, P, -\frac{3P}{5}\right)$$

$$\vec{R} = \vec{S}_{AB} + \vec{S}_{AC} = \left(\frac{4P}{5}, 0, -\frac{3P}{5}\right) + \left(\frac{4P}{5}, P, -\frac{3P}{5}\right)$$

$$\vec{R} = \left(\frac{8P}{5}, P, -\frac{6P}{5}\right) \quad R = \sqrt{\left(\frac{8P}{5}\right)^2 + P^2 + \left(\frac{6P}{5}\right)^2}$$

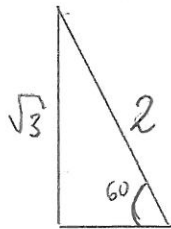
2,20 forts.

$$R = \sqrt{\frac{64P^2}{25} + P^2 + \frac{36P^2}{25}} = \sqrt{\frac{64P^2 + 25P^2 + 36P^2}{25}}$$

$$R = \sqrt{\frac{125P^2}{25}} = \underline{\underline{P \cdot \sqrt{5}}}$$

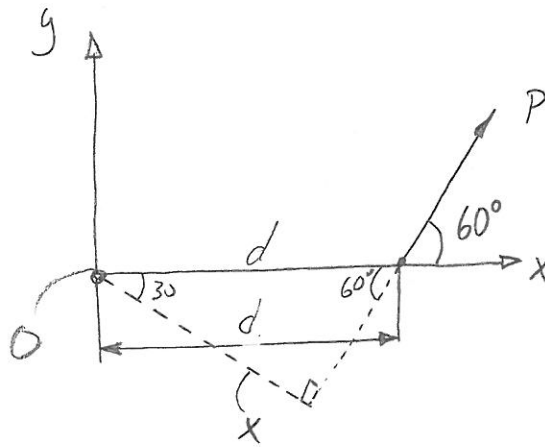
Svar: Resultatet är $P \cdot \sqrt{5}$

2.26) a)

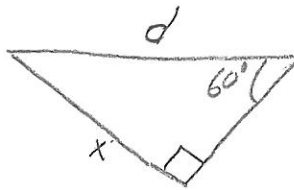


$$\sin 60 = \frac{\sqrt{3}}{2}$$

$$\cos 60 = \frac{1}{2}$$



hävärmen till kraften P mao.



$$\sin 60 = \frac{x}{d}$$

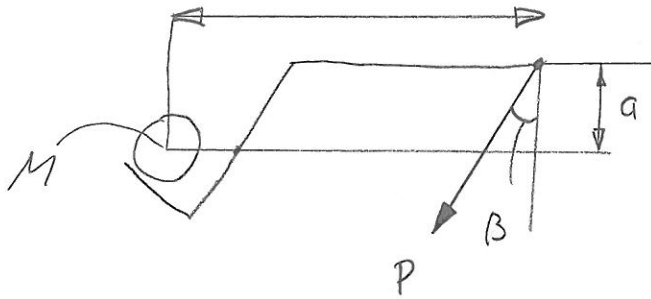
$$\Rightarrow x = d \sin 60^\circ$$

$$x = \frac{d \cdot \sqrt{3}}{2}$$

$$\vec{M}_0 = P \cdot \frac{d \cdot \sqrt{3}}{2}$$

$$\vec{M}_0 = \frac{\sqrt{3} P \cdot d}{2} \vec{e}_2$$

2.29

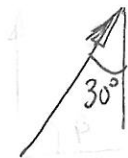


Hur stort moment bidrar P med i M_A

$$\vec{M}_A = P \cdot \cos \beta \cdot b - P \cdot \sin \beta \cdot a$$

2.30] P kraftens vridande förmåga

är y komponenten riktad uppåt



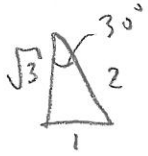
$$\beta = 30$$



$$\sin \beta = \frac{P_y}{P} \Rightarrow P_y = P \cdot \cos \beta = \frac{\sqrt{3} P}{2}$$

$$\vec{M}_O = \frac{\sqrt{3} P}{2} \cdot r = \frac{\sqrt{3} \cdot 60}{20} \cdot 0,08 = \underline{\underline{4,16 \text{ Nm}}}$$

2.32



P :s vridande förmåga map O .

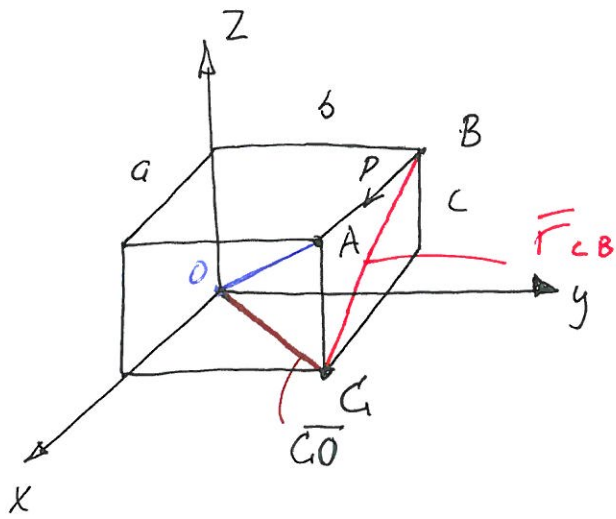
$$\vec{M}_O = P \cdot \cos\beta \cdot d \cos\alpha - P \cdot \sin\beta \cdot d \sin\alpha$$

$$\begin{aligned} \alpha &= 30^\circ \\ \beta &= 30^\circ \end{aligned} \quad \vec{M}_O = \frac{2 \cdot \sqrt{3}}{2} \cdot \frac{0,025 \cdot \sqrt{3}}{2} - \frac{2 \cdot 1}{2} \cdot \frac{0,025 \cdot 1}{2}$$

$$\vec{M}_O = \frac{3 \cdot 0,025}{2} - \frac{0,025 \cdot 1}{2} = \frac{2 \cdot 0,025}{2} \text{ Nm}$$

$$\vec{M}_O = 0,025 \text{ Nm}$$

2.37



$$\bar{P} = P \cdot \hat{e}_{BA}$$

$$\hat{e}_{BA} = \frac{\bar{r}_{BA}}{|\bar{r}_{BA}|} = \frac{(a, b, c) - (0, b, c)}{\sqrt{a^2}} = \frac{a(1, 0, 0)}{a} = (1, 0, 0)$$

$$\bar{P} = P \cdot (1, 0, 0) = (P, 0, 0)$$

$$\bar{M}_O = \bar{r} \times \bar{P} = (a, b, c) \times (P, 0, 0)$$

$$= \begin{vmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ a & b & c \\ P & 0 & 0 \end{vmatrix} = (0, c \cdot P, -b \cdot P) = c \cdot P \hat{e}_y - b \cdot P \hat{e}_z$$

$\bar{M}_B = \bar{0}$ för \bar{P} vektorn har ingen "moment arm" i punkten B.

$$\bar{M}_C = \bar{M}_O + \bar{r}_{CO} \times \bar{P}$$

$$\bar{r}_{CO} = (-a, -b, 0)$$

jämför ekv 2.15 sid 37

(spets - fotpunkt)
origo punkt C

$$\bar{r}_{CO} \times \bar{P} = \begin{vmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ -a & -b & 0 \\ P & 0 & 0 \end{vmatrix} = (0, 0, b \cdot P)$$

⇒ forts.

2.37 forts

$$\bar{M}_c = (0, c \cdot P, -b \cdot P) + (0, 0, b \cdot P)$$

$$\bar{M}_c = (0, cP, 0)$$

Alternativ lösning:

$$\bar{M}_c = \bar{r}_{cB} \times \bar{P}$$

jämför ekvation 2.9
sidan 23
 $\bar{M}_A = \bar{r}_{AP} \times \bar{F}$
A = den punkt vi vill räkna moment.
kraftens angreppspunkt.

$$\bar{r}_{cB} = (0, b, c) - (a, b, 0) = (-a, 0, c)$$

$$\bar{M}_c = (-a, 0, c) \times (P, 0, 0) = \begin{vmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ -a & 0 & c \\ P & 0 & 0 \end{vmatrix}$$

$$= (0 \cdot 0 - c \cdot 0, -(-a \cdot 0 - c \cdot P), -a \cdot 0 - 0 \cdot P) = (0, cP, 0)$$

Momentet kring punkten C kan alltså lösas på två sätt

Alt Lösung igen:

$$2,37 \quad \overline{M}_c = \overline{r}_c \times \overline{p}$$

$$\overline{p} = (p, 0, 0)$$

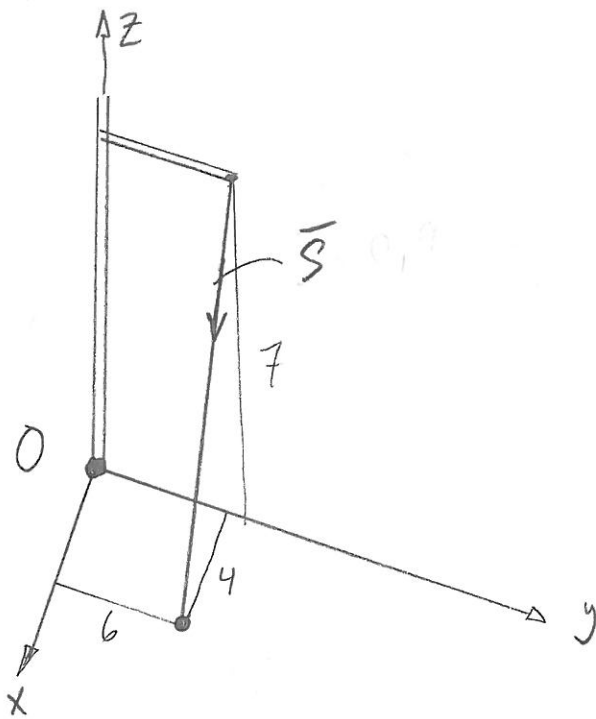
$$\overline{r}_{CA} = (a, b, c) - (a, b, 0) = (0, 0, c)$$

$$\overline{M}_c = \overline{r}_{CA} \times \overline{p} = \begin{vmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ 0 & 0 & c \\ p & 0 & 0 \end{vmatrix} = (0, c \cdot p, 0)$$

$$\overline{M}_c =$$

2,39

$$S = 0,9 \text{ kN}$$



$$\boxed{\bar{M}_0 = \bar{r} \times \bar{F}}$$

M_0 söks

$$\bar{F} = \bar{S} = S \cdot \hat{e}_{AB} \Rightarrow$$

$$\hat{e}_{AB} = \frac{(4, 6, 0) - (0, 2, 7)}{\sqrt{16+16+49}} = \frac{4, 4, -7}{9} = \frac{1}{9}(4, 4, -7)$$

$$\hat{e}_{AB} = \frac{(4, 4, -7)}{9} \Rightarrow \bar{F} = \frac{900}{9}(4, 4, -7) = (400, 400, -700) \text{ N}$$

$$\bar{r}_A = (0, 2, 7) \text{ m}$$

$$\bar{M}_0 = \bar{r} \times \bar{F} = (0, 2, 7) \times (400, 400, -700) \text{ Nm}$$

$$= \begin{vmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ 0 & 2 & 7 \\ 4 & 4 & -7 \end{vmatrix} \text{ Nm} = 100(-42\hat{e}_x, 28\hat{e}_y, -8\hat{e}_z) \text{ Nm}$$

$$= 200(-21\hat{e}_x, 14\hat{e}_y, -4\hat{e}_z) \text{ Nm}$$

2.48)

$$\tan \beta = \frac{3}{4}$$

$$\sin \beta = \frac{3}{5}$$

$$\cos \beta = \frac{4}{5}$$

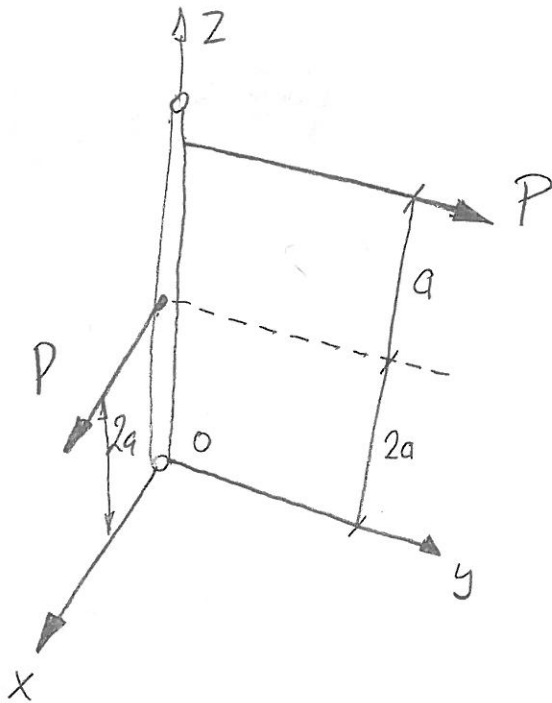
$$\rightarrow x: 250 \cdot \frac{3}{5} - 100 = 50 \text{ N}$$

$$\uparrow y: 250 \cdot \frac{4}{5} - 100 = 100$$

$$\vec{F} = (50, 100, 0) \text{ N}$$

$$\vec{M}_O = (0, 0, 60) \text{ Nm}$$

2.52)



$$x: \downarrow = P - 0 = P$$

$$y: \downarrow = P - 0 = P$$

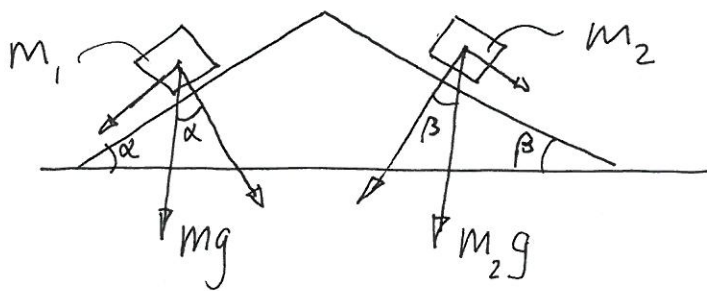
$$\vec{M}_x = -P \cdot 3a = -P \cdot 3a \hat{e}_x$$

$$\vec{M}_y = 2a \cdot P = P \cdot 2a \hat{e}_y$$

$$\vec{F} = P \hat{e}_x + P \hat{e}_y$$

$$\vec{M} = -3Pa \hat{e}_x + 2Pa \hat{e}_y$$

3,1]



$m_1 = m$ är givet sökt m_2

Lådorna i jämvikt ger:

$$mg \cdot \sin \alpha = m_2 \cdot g \cdot \sin \beta$$

$$m_2 = \frac{m \cdot \sin \alpha}{\sin \beta}$$

3.2]



$$S \cdot \cos \alpha \cdot L \sin \beta + \underbrace{S \sin \alpha \cdot L \cos \beta}_{S \cdot \cos \beta} - \underbrace{mg \cdot \frac{L}{2} \cdot \sin \beta}_{\text{h\u00e4varm}} = 0$$

$$S = \frac{mg \cdot \sin \beta}{2 (\cos \alpha \cdot \sin \beta + \sin \alpha \cdot \cos \beta)}$$

$$S = \frac{mg \cdot \frac{\sqrt{2}}{2}}{2 \left(\frac{3}{5} \cdot \frac{\sqrt{2}}{2} + \frac{4}{5} \cdot \frac{\sqrt{2}}{2} \right)} = \frac{mg \cdot \frac{\sqrt{2}}{2}}{\frac{3 \cdot \sqrt{2}}{5} + \frac{4 \cdot \sqrt{2}}{5}}$$

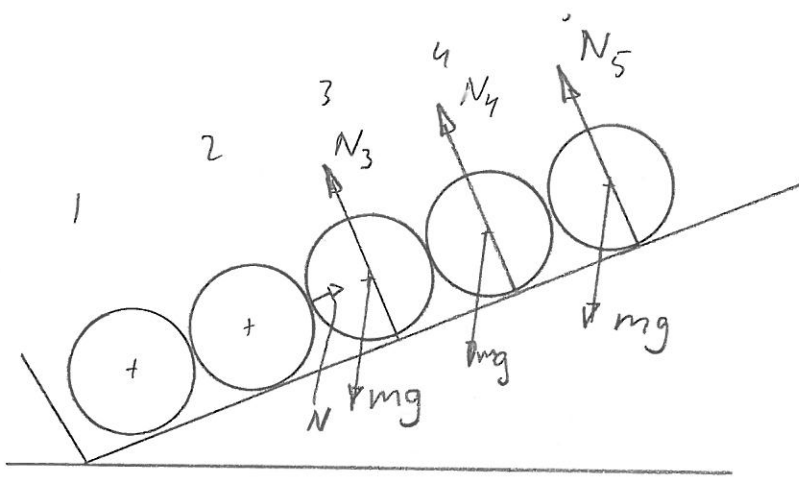
$$= \frac{mg \cdot \frac{\sqrt{2}}{2} \cdot 5}{3 \cdot \sqrt{2} + 4 \sqrt{2}} = \frac{mg \cdot 5}{\frac{2}{\sqrt{2}} (3 \cdot \sqrt{2} + \sqrt{2} \cdot 4)}$$

$$= \frac{mg \cdot 5}{6 + 8} = \frac{5}{14} mg$$

$$\beta = \frac{\pi}{4}$$

$$\tan \alpha = \frac{4}{3}$$

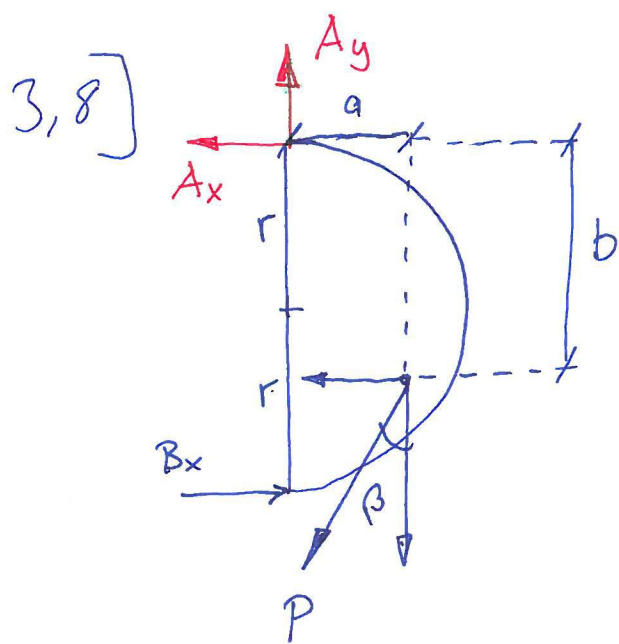
3,7]



Jämvikt för det frilagda systemet

$$\nearrow : N - 3mg \sin \beta = 0$$

$$\Rightarrow N = 3mg \sin \beta$$



ſökt A_x e A_y

$$\begin{cases} \Sigma \bar{F} = 0 \\ \Sigma \bar{M} = 0 \end{cases} \Rightarrow \begin{aligned} \rightarrow : B_x - P \sin \beta - A_x &= 0 \quad (\text{I}) \\ \uparrow : A_y - P \cdot \cos \beta &= 0 \\ \Rightarrow A_y &= P \cdot \cos \beta \end{aligned}$$

$$\hat{A} : B_x \cdot 2r - P \cdot \sin \beta \cdot b - P \cdot \cos \beta \cdot a = 0$$

$$B_x = \frac{P \cdot (b \cdot \sin \beta + a \cos \beta)}{2r} \quad (\text{III}) \quad \text{ins i (I)}$$

$$\frac{P \cdot b \cdot \sin \beta}{2r} + \frac{P \cdot a \cdot \cos \beta}{2r} - P \cdot \sin \beta - A_x = 0$$

$$A_x = \frac{P}{2r} (b \cdot \sin \beta + a \cdot \cos \beta - 2r \cdot \sin \beta)$$

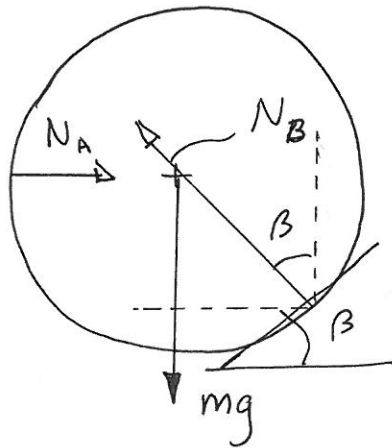
$$A_x = \frac{P}{2r} ((b - 2r) \sin \beta + a \cos \beta)$$

$$\text{Svar: } A_x = \frac{P}{2r} ((b - 2r) \sin \beta + a \cos \beta)$$

$$A_y = P \cos \beta$$

3,10

Lösning frilägg sätt ut krafter



$$\rightarrow : N_A = N_B \sin \beta = 0 \quad \text{I}$$

$$\uparrow : N_B \cdot \cos \beta - mg = 0 \quad \text{II}$$

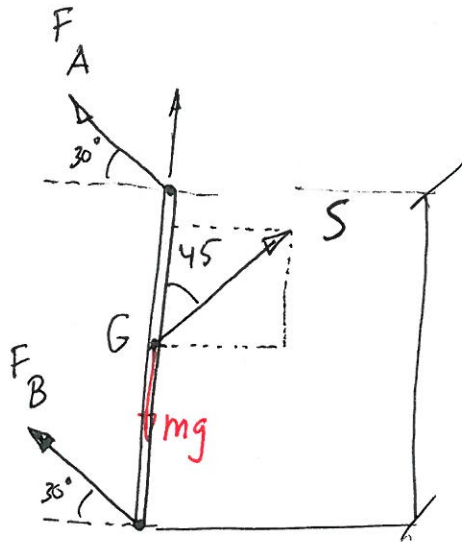
$$N_B = \frac{mg}{\cos \beta} \quad \text{ins i (I)} \Rightarrow$$

$$N_A = \frac{mg}{\cos \beta} \cdot \sin \beta = mg \tan \beta$$

Svar: $N_A = mg \tan \beta$

$$N_B = \frac{mg}{\cos \beta}$$

3,12



$$\begin{array}{c} \sqrt{3} \\ \triangle \\ 1 \end{array} \quad \cos 30 = \frac{\sqrt{3}}{2}$$

$$\rightarrow S \cdot \frac{\sqrt{2}}{2} - F_A \cdot \cos 30 - F_B \cdot \cos 30 = 0 \quad I$$

$$\uparrow : F_A \cdot \sin 30 + F_B \cdot \sin 30 + \frac{S \cdot \sqrt{2}}{2} - mg = 0 \quad II$$

$$\left(\overset{A}{B} : \frac{S \cdot \sqrt{2}}{2} \cdot \frac{d}{2} - d \cdot F_A \cdot \cos 30 = 0 \right)$$

~~at B:~~

$$\overset{G}{G} : \frac{d}{2} \cdot \frac{\sqrt{3} F_A}{2} - \frac{d}{2} \cdot \frac{\sqrt{3} F_B}{2} = 0 \quad III$$

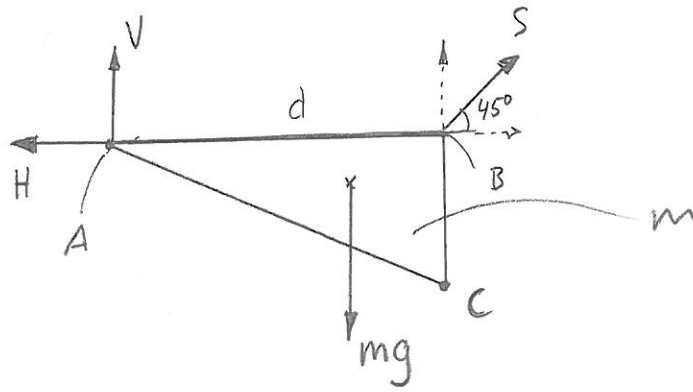
$$F_A = F_B \Rightarrow i \quad I$$

$$\frac{S \cdot \sqrt{2}}{2} - F_A \cdot \frac{\sqrt{3}}{2} - F_A \cdot \frac{\sqrt{3}}{2} = 0 \Rightarrow F_A = \frac{\sqrt{2}}{2 \cdot \sqrt{3}} S$$

$$\text{ins i II} \quad \frac{\sqrt{2}}{2 \cdot \sqrt{3}} S + \frac{\sqrt{2}}{2} S - mg = 0$$

$$S = \frac{mg \cdot 2 \cdot \sqrt{3}}{\sqrt{2} + \sqrt{6}}$$

3,13]



$$\rightarrow: S \cdot \cos 45 - H = 0$$

$$\uparrow: S \cdot \sin 45 + V - mg = 0$$

$$\widehat{B}: V \cdot d - \frac{d}{3} \cdot mg = 0$$

$$V = \frac{mg}{3} \quad \text{ins i 2}$$

$$S \cdot \frac{1}{\sqrt{2}} = mg - \frac{mg}{3} \quad 2 \cdot \sqrt{2}$$

$$S = \frac{\sqrt{2} \cdot 2mg}{3} = \frac{2 \cdot \sqrt{2} mg}{3} \quad \text{ins i 1}$$

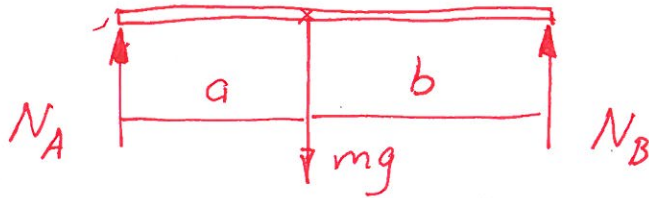
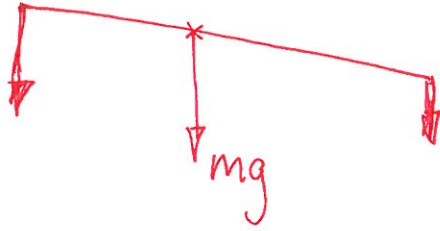
$$1 \Rightarrow \frac{2 \cdot \sqrt{2} \cdot mg}{3} \cdot \frac{1}{\sqrt{2}} - H = 0$$

$$H = \frac{2mg}{3}$$

$$\text{Svar: } V = \frac{mg}{3} \quad H = \frac{2mg}{3} \quad S = \frac{2 \cdot \sqrt{2} mg}{3}$$

3,14

$$b = 3a$$



$$\uparrow: N_A + N_B - mg = 0 \quad \text{I}$$

$$\overset{\curvearrowright}{A}: mg \cdot a - N_B(a+b) = 0 \quad \text{II}$$

$$N_B = \frac{mga}{a+b} \quad \text{ins i I} \Rightarrow$$

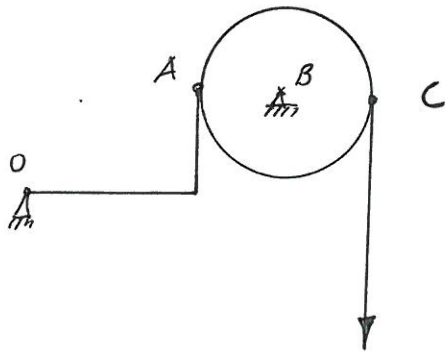
$$N_A = mg - \frac{mga}{a+b} = \frac{mga + mgb}{a+b} - \frac{mga}{a+b} = \frac{mgb}{a+b}$$

$$b = 3a \Rightarrow$$

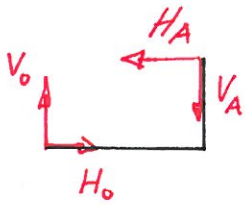
$$N_A = \frac{mg \cdot 3a}{a+3a} = \frac{\cancel{3}mg}{4\cancel{a}} = \underline{\underline{\frac{3mg}{4}}}$$

$$N_B = \frac{mga}{4a} = \underline{\underline{\frac{mg}{4}}}$$

3,18



Lösning frilägg delarna och sätt ut krafter

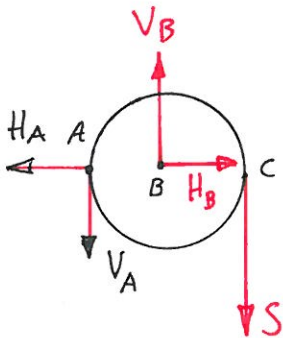


$$\rightarrow : H_0 - H_A = 0$$

$$\uparrow : V_0 - V_A = 0$$

$$\curvearrow : V_A \cdot 2r - H_A \cdot r = 0$$

$$V_A = \frac{H_A}{2} \Rightarrow H_A = 2V_A$$



$$\rightarrow : -H_A + H_B = 0 \quad \Delta$$

$$\downarrow : V_A - V_B + S = 0 \quad \star$$

$$\curvearrow : S \cdot 2r - V_B \cdot r = 0$$

$$V_B = \frac{2S \cdot r}{r} = 2S \quad \text{ins i } \star$$

$$V_A - 2S + S = 0$$

$$V_A = S$$

$$H_A = 2V_A = 2S \quad \text{ins i } \Delta \Rightarrow H_B = H_A = 2S$$

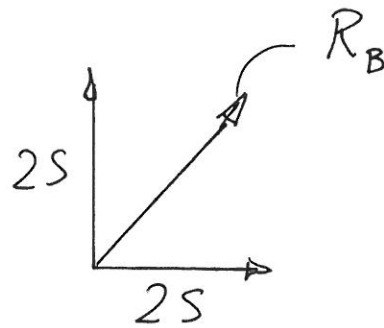
\Rightarrow
Forts

3,18 forts.

⇒

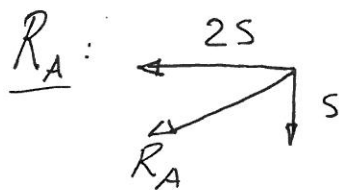
$$H_B = 2S$$

$$V_B = 2S$$



$$R_B = \sqrt{(2S)^2 + (2S)^2} = \sqrt{8S^2} = \sqrt{4} \cdot \sqrt{2} \cdot S$$

$$R_B = \underline{\underline{2 \cdot \sqrt{2} S}}$$



$$R_A = \sqrt{(2S)^2 + S^2} = \underline{\underline{\sqrt{5} S}}$$

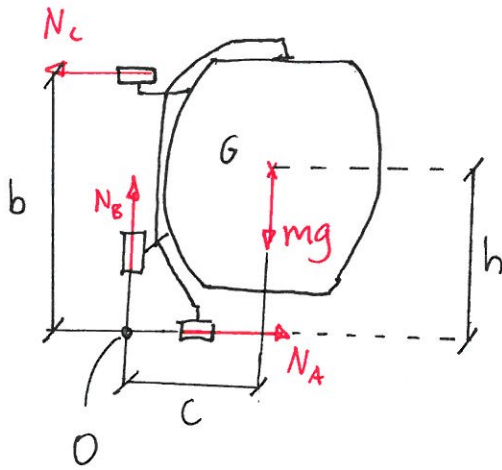
Svar: $R_A = \sqrt{5} \cdot S$

$$R_B = 2 \cdot \sqrt{2} S$$

3,21

Massan m

$$\text{tyngd} = mg$$



lösning frilägg sätt ut krafter.

$$\rightarrow : N_A - N_C = 0$$

$$\uparrow : N_B - mg = 0$$

$$\odot : mg \cdot c - N_C \cdot b = 0$$

$$N_B = mg \quad N_A = N_C$$

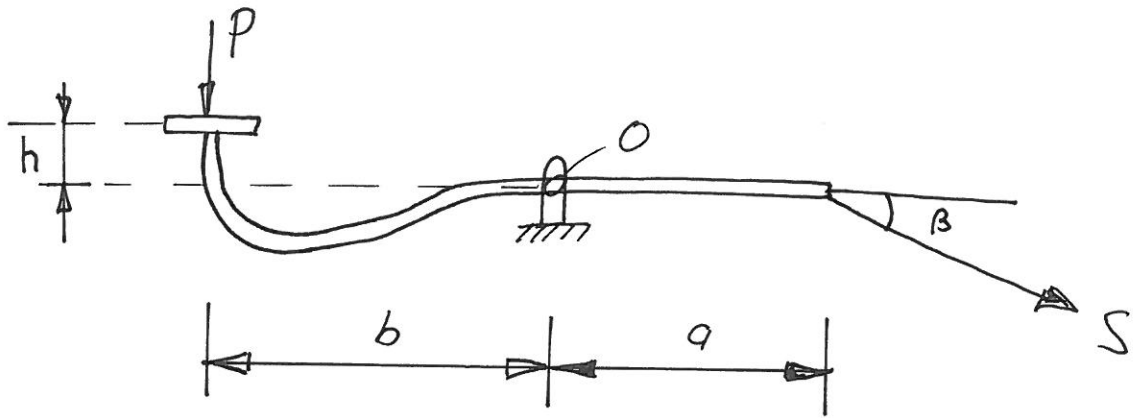
$$N_C = \frac{m \cdot g \cdot c}{b}$$

$$N_A = mg \frac{c}{b}$$

Eftersom det var två par hjul i varje punkt så är reaktionskrafterna hälften så stora.

$$N_B = \frac{mg}{2} \quad , \quad N_A = N_C = \frac{c}{2b} mg$$

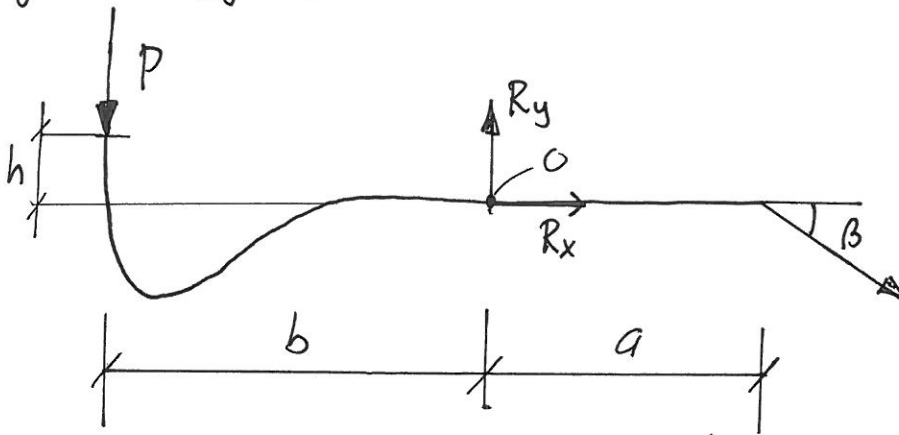
3,33



Lösning: Frilägg systemet och sätt ut krafter

Sökt: S ?

R_x
 R_y } R



$$\rightarrow: S \cdot \cos \beta + R_x = 0 \quad (1) \Rightarrow R_x = -S \cdot \cos \beta$$

$$\uparrow: R_y - P - S \cdot \sin \beta = 0 \quad (2) \Rightarrow R_y = P + S \cdot \sin \beta$$

$$\circlearrowleft: S \cdot \sin \beta \cdot a - P \cdot b = 0 \quad (3)$$

$$S = \frac{P \cdot b}{a \cdot \sin \beta} \quad \text{ins i (1) o (2)}$$

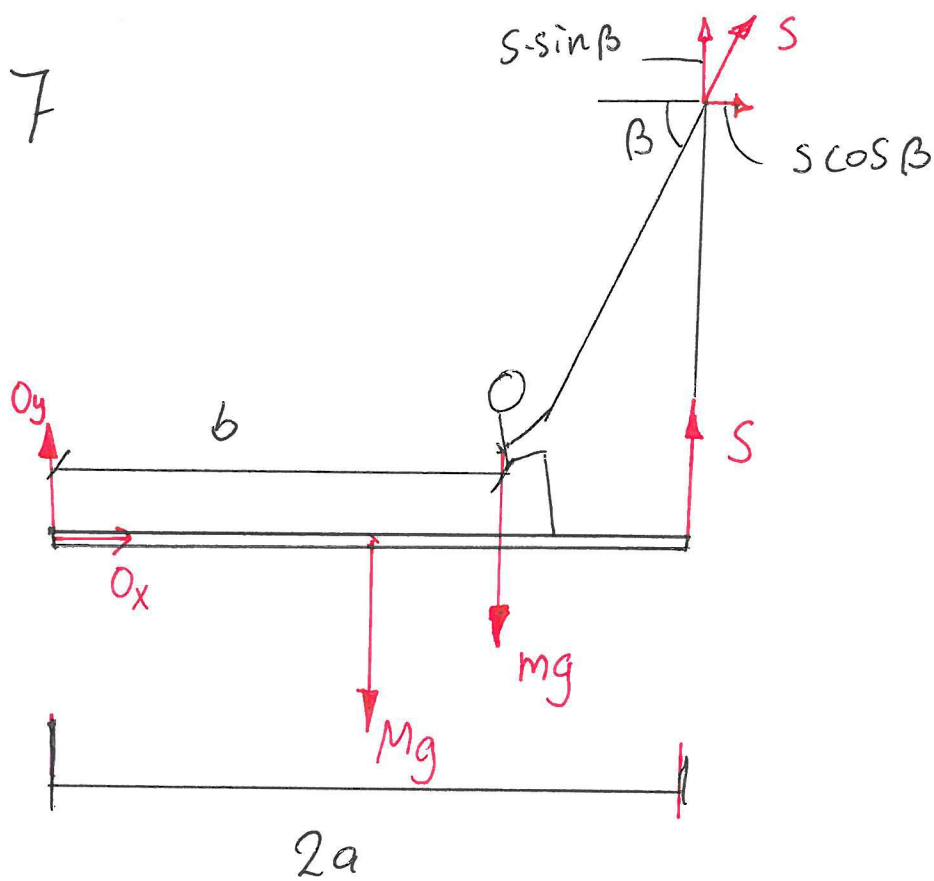
$$R_x = -\frac{P \cdot b}{a \cdot \sin \beta} \cdot \cos \beta = -\frac{b}{a \tan \beta} P$$

$$R_y = P + \frac{P \cdot b \cdot \sin \beta}{a \cdot \sin \beta} = P + \frac{Pb}{a} = \frac{P(a+b)}{a}$$

$$R = \sqrt{R_x^2 + R_y^2} = P \sqrt{\frac{(a+b)^2}{a^2} + \frac{b^2}{a^2 \tan^2 \beta}} = \frac{P}{a} \sqrt{(a+b)^2 + \frac{b^2}{\tan^2 \beta}}$$

$$\text{Svar: } S = \frac{P \cdot b}{a \sin \beta} \quad R = \frac{P}{a} \sqrt{(a+b)^2 + \frac{b^2}{\tan^2 \beta}}$$

3.37
(xxx)



Momentekvation kring O ger oss direkt S:

$$\curvearrowright O: S \cdot 2a + S \cdot \sin \beta \cdot 2a - S \cdot \cos \beta \cdot h - Mg a - mg b = 0$$

$$S \cdot 2a + S \cdot \sin \beta \cdot 2a - S \cdot \cos \beta \cdot h = Mg a + mg b$$

$$S = \frac{(M a + m b) g}{(2a + 2a \sin \beta) - \cos \beta \cdot h}$$

Kraftjämvikt: $\rightarrow O_x + S \cos \beta = 0$

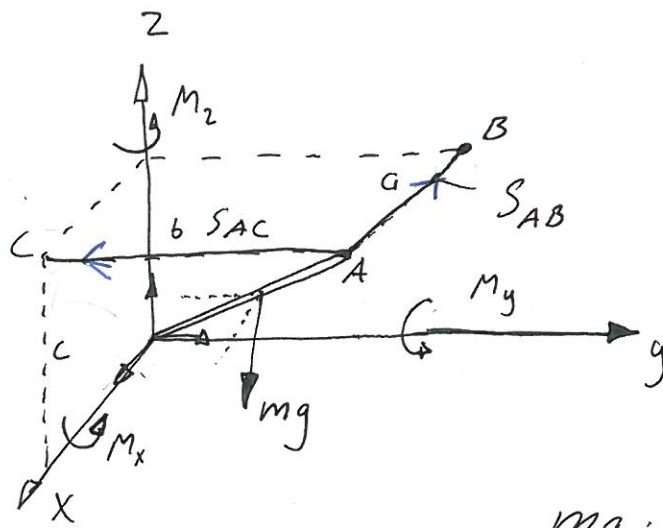
$$O_x = -S \cos \beta$$

$$\uparrow: O_y + S + S \cdot \sin \beta - Mg - mg = 0$$

$$O_y = (M + m)g - S - S \cdot \sin \beta = (M + m)g - S(1 + \sin \beta)$$

Svar: $S = \frac{(M a + m b) g}{2a(1 + \sin \beta) - h \cdot \cos \beta}$, $O_x = -S \cos \beta$
 $O_y = (M + m)g - S(1 + \sin \beta)$

3,59



mg is ligg $(\frac{a}{2}, \frac{b}{2}, \frac{c}{2})$

$$\boxed{\Sigma \vec{F} = 0}$$

Bestäm S_{AB} o S_{AC}

$$\left. \begin{array}{l} \Sigma F_x = 0 \Rightarrow -S_{AB} + R_x = 0 \\ \Sigma F_y = 0 \Rightarrow -S_{AC} + R_y = 0 \\ \Sigma F_z = 0 \Rightarrow R_z - mg = 0 \end{array} \right\} \begin{array}{l} 1 \\ 2 \\ 3 \end{array}$$

$$\Sigma \vec{M}_o = 0$$

$$\Sigma M_{ox} = S_{AC} \cdot c - mg \cdot \frac{b}{2} = 0 \quad 4$$

$$\Sigma M_{oy} = -S_{AB} \cdot c + mg \cdot \frac{a}{2} = 0 \quad 5$$

$$\Sigma M_{oz} = -S_{AC} \cdot a + S_{AB} \cdot b = 0 \quad 6$$

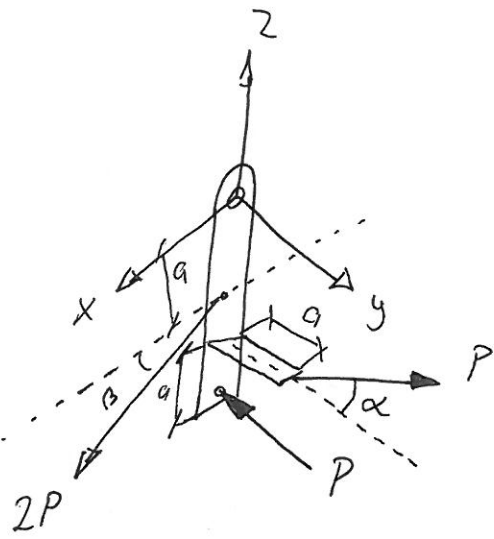
$$\text{och } 4 \text{ o } 5 \Rightarrow S_{AC} = \frac{mg b}{2c}$$

$$S_{AB} = \frac{mg a}{2c}$$

Ekv 6 behövs ej men man kan kontrollera sin lösning mha ett
sätta in $S_{AC} = \frac{mg b}{2c} \Rightarrow -\frac{mg b \cdot a}{2c} + S_{AB} \cdot b = 0$

$$S_{AB} \cdot b = \frac{mg b a}{2c b} = \frac{mg a}{2c} = \frac{mg a}{2c} \quad \text{Vilket stämmer.}$$

3,64



↗ : x led $2P \cdot \cos \beta - P \sin \alpha$

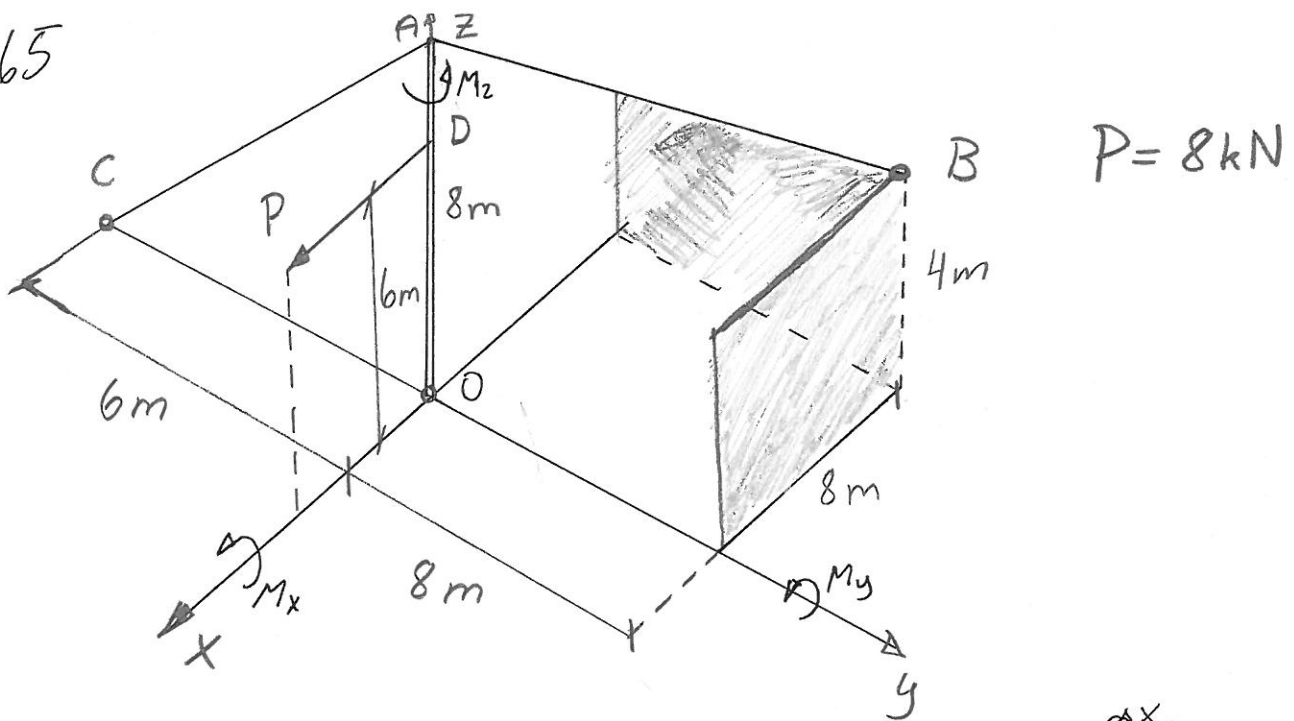
↘ : y led : $P \cos \alpha - P$

↑ : z led - $2P \sin \beta$

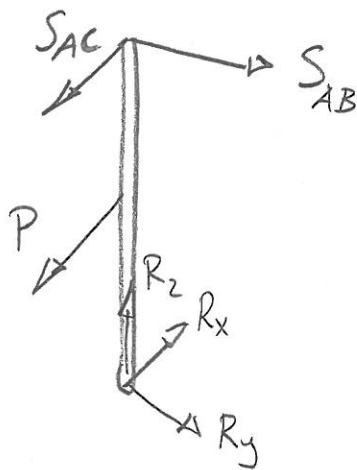
$$\vec{R} = (2P \cdot \cos \beta - P \sin \alpha) \hat{e}_x + (P \cos \alpha - P) \hat{e}_y - 2P \sin \beta \hat{e}_z$$

$$\vec{R} = (2P \cos \beta - P \sin \alpha, P \cos \alpha - P, -2P \sin \beta)$$

3,65



$P = 8 \text{ kN}$



Kraft ekvationerna

$$\sum \vec{F} = \vec{0}$$

ger att 3 ekv 5 obekanta

Moment ekvationerna

$$\sum \vec{M}_O = \vec{0} \Rightarrow 3 \text{ ekvationer och } 2 \text{ obekanta alltså kan vi}$$

Teckna S_{AB} som en vektor

$$\vec{S}_{AB} = S_{AB} \cdot \hat{e}_{AB}$$

$$\hat{e}_{AB} = \frac{\vec{AB}}{|\vec{AB}|} = \frac{(-8, 8, 4) - (0, 0, 8)}{|(-8, 8, 4) - (0, 0, 8)|} = \frac{(-8, 8, -4)}{|(-8, 8, -4)|}$$



3,65 forts

$$= \frac{4(-2, 2, -1)}{\sqrt{(-8)^2 + 8^2 + (-4)^2}} = \frac{4(-2, 2, -1)}{\sqrt{64 + 64 + 16}} = \frac{4(-2, 2, -1)}{12}$$

$$= \frac{1}{3}(-2, 2, -1)$$

$$\bar{S}_{AB} = \frac{S_{AB}}{3}(-2, 2, -1) N$$

$$\hat{e}_{AC} = \frac{\vec{AC}}{|\vec{AC}|} = \frac{\vec{r}_C - \vec{r}_A}{|\vec{r}_C - \vec{r}_A|} = \frac{(0, -6, 0) - (0, 0, 8)}{|(0, -6, 0) - (0, 0, 8)|}$$

$$= \frac{(0, -6, -8)}{\sqrt{(-6)^2 + (-8)^2}} = \frac{(0, -6, -8)}{\sqrt{36 + 64}} = \frac{2(0, -3, -4)}{10}$$

$$= \frac{(0, -3, -4)}{5}$$

$$\bar{S}_{AC} = \frac{S_{AC}}{5}(0, -3, -4) N$$

$$\sum \bar{M}_x = 0 \Rightarrow -\frac{S_{AB} \cdot 2}{3} \cdot 8 + \frac{S_{AC} \cdot 3}{5} \cdot 8 = 0 \Rightarrow S_{AB} = \frac{9S_{AC}}{10}$$

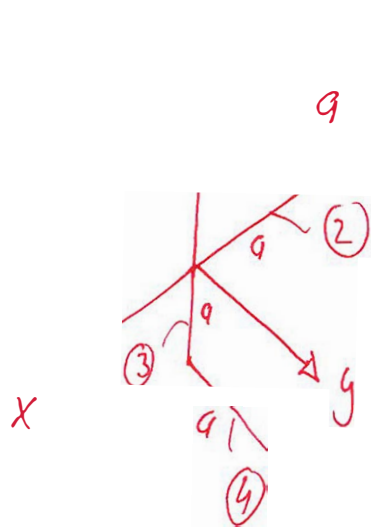
$$\sum \bar{M}_y = 0 \Rightarrow P \cdot 6 - \frac{2S_{AB} \cdot 8}{3} = 0$$

$$S_{AB} = \frac{6P \cdot 3}{2 \cdot 8} = \frac{18P}{16} = \frac{9P}{8} \Rightarrow$$

$$P = 8 \text{ kN}$$

$$S_{AC} = \frac{10 \cdot S_{AB}}{9} = \frac{10 \cdot 9P}{8 \cdot 8} = \frac{10P}{8}$$

$$\Rightarrow S_{AB} = 9 \text{ kN} \quad S_{AC} = 10 \text{ kN}$$



$$(l_1 + l_2 + l_3 + l_4) \cdot X_G$$

$$l_1 \cdot (-a) + l_2 \cdot \left(-\frac{a}{2}\right) + 0 + 0$$

$$4a X_G = a \cdot (-a) + a \cdot \left(-\frac{a}{2}\right)$$

$$X_G = -\frac{3a}{8} \quad 4a$$

$$4a \cdot y_G = 0 + 0 + 0 + \frac{a}{2} \cdot a$$

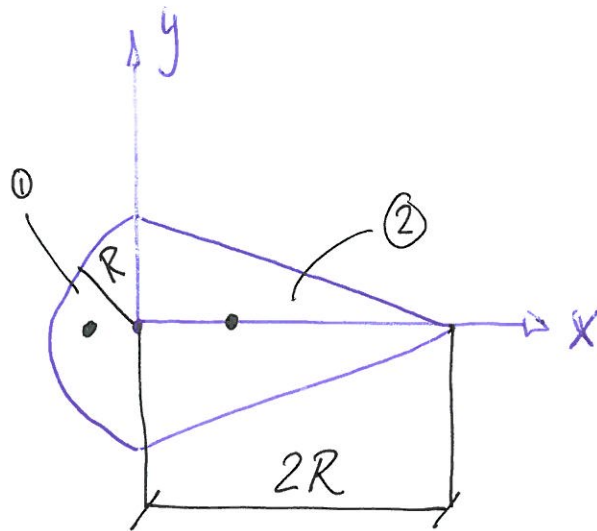
$$y_G = \frac{a}{8}$$

$$4a z_G = \cancel{a \cdot \frac{a}{2}} + 0 + \cancel{a \cdot \left(-\frac{a}{2}\right)} + a \cdot (-a)$$

$$= -\frac{a^2}{4}$$

$$\vec{F}_G = \left(-\frac{3a}{8} \mid \frac{a}{8} \mid \frac{-a}{4} \right)$$

4,2)



Pga symmetri ligger TP på x-axeln.

$$A_1 = \frac{\pi \cdot R^2}{2}$$

$$A_2 = \frac{2R \cdot 2R}{2}$$

$$(A_1 + A_2) X_G = A_2 \cdot X_2 - A_1 \cdot X_1$$

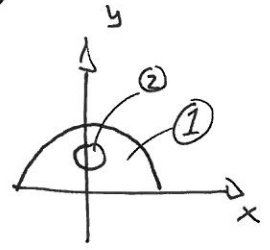
$$\left(\frac{\pi \cdot R^2}{2} + \frac{4R^2}{2} \right) X_G = \frac{4R^2 \cdot 2R}{2 \cdot 3} - \frac{\pi \cdot R^2}{2} \cdot \frac{4R}{3\pi}$$

$$X_G = \frac{\frac{8R}{3} - \frac{4R}{3}}{\pi + 4} = \frac{4R}{3(\pi + 4)}$$

4,3)

$$A_1 = \frac{\pi \cdot (6a)^2}{2} = \frac{36\pi a^2}{2} = 18a^2\pi$$

$$A_2 = \pi \cdot (2a)^2 = 4a^2\pi$$



$$(A_1 - A_2) \cdot y_6 = A_1 \cdot y_1 - A_2 \cdot y_2$$

$$\pi a^2 (18 - 4) \cdot y_6 = 18\pi \cdot a^2 \cdot \frac{4 \cdot 6a}{3\pi} - 4a^2 \cdot 3a \cdot \pi$$

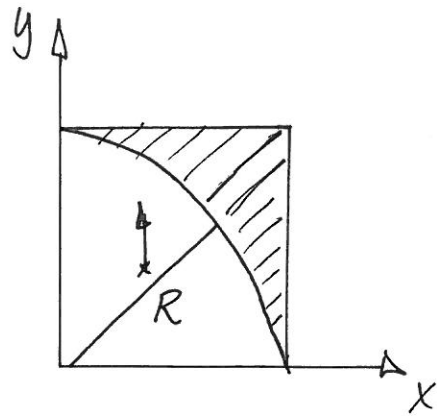
$$14\pi a^2 y_6 = \frac{a^2 18\pi \cdot 4 \cdot 6a}{3\pi} - a^2 \cdot 12a \cdot \pi$$

$$14\pi y_6 = 144a - 12\pi a$$

$$y_6 = \frac{12\pi a \left(\frac{12}{\pi} - 1 \right)}{14\pi} = \frac{6a \left(\frac{12}{\pi} - 1 \right)}{7}$$

$$\text{Svar: } y_6 = \frac{6a}{7} \left(\frac{12}{\pi} - 1 \right) \approx \underline{\underline{2,4a}}$$

4,4

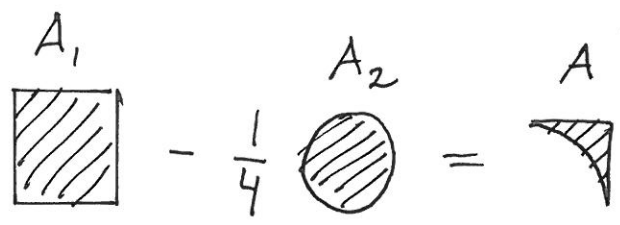


$$A_1 = R^2$$

$$A_2 = \pi R^2$$

$$x_1 = R/2 = y_1$$

$$x_2 = \frac{4R}{3\pi} = y_2$$



$$\left(R^2 - \frac{\pi R^2}{4}\right) x_G = R^2 \cdot \frac{R}{2} - \frac{\pi R^2}{4} \cdot \frac{4R}{3\pi}$$

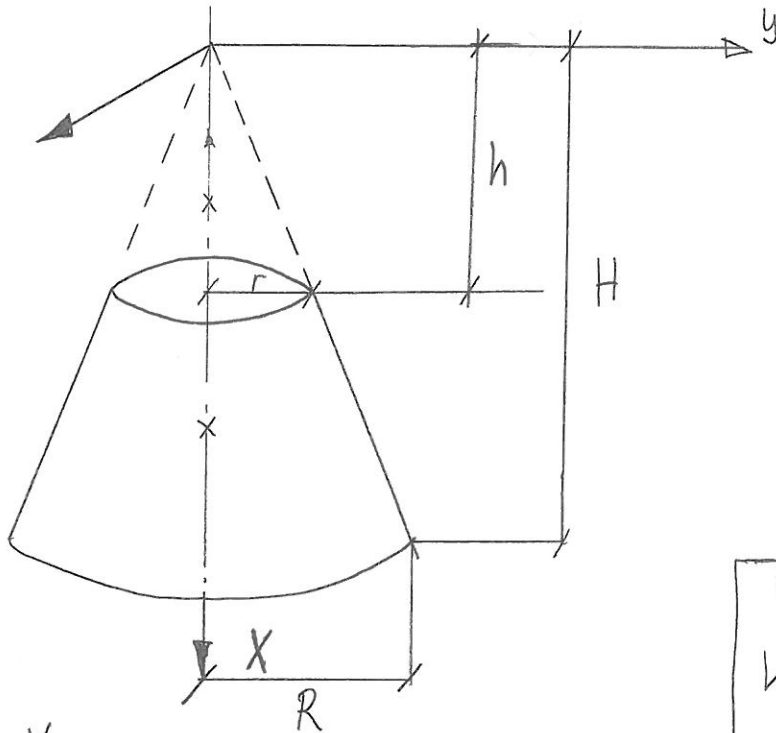
$$R^2 \left(1 - \frac{\pi}{4}\right) x_G = \frac{R^3}{2} - \frac{\cancel{\pi} R^3}{\cancel{4} \cdot 3}$$

$$\left(1 - \frac{\pi}{4}\right) x_G = \frac{R}{2} - \frac{R}{3}$$

$$x_G = \frac{R}{6} / \left(1 - \frac{\pi}{4}\right) = \frac{R}{6 \left(\frac{4-\pi}{4}\right)} = \frac{4R}{6(4-\pi)} = \frac{2R}{3(4-\pi)}$$

$$x_G = \frac{2R}{12-3\pi} \quad \text{Symmetri ger } y_G = \frac{2R}{12-3\pi}$$

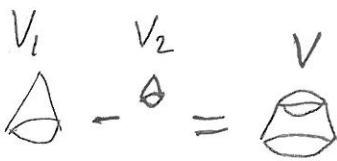
4,5



likformighet.

$$\left[\frac{h}{r} = \frac{H}{R} \right] \Rightarrow r = \frac{h \cdot R}{H}$$

Volym rät cirkulär
kon $\frac{\pi r^2 \cdot h}{3}$
 $X_G = \frac{3h}{4}$



$$V_1 = \frac{\pi \cdot R^2 \cdot H}{3} \quad X_1 = \frac{3H}{4}$$

$$V_2 = \frac{\pi \cdot r^2 \cdot h}{3} = \frac{\pi \cdot \left(\frac{h \cdot R}{H}\right)^2 \cdot h}{3} \quad X_2 = \frac{3h}{4}$$

$$(V_1 - V_2) X_G = V_1 \cdot X_1 - V_2 \cdot X_2$$

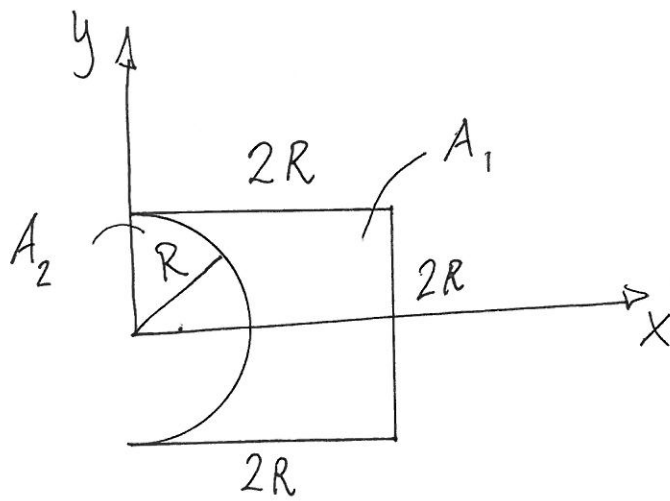
$$\left(\frac{\pi \cdot R^2 \cdot H}{3} - \frac{\pi \cdot \frac{h^2 \cdot R^2}{H^2} \cdot h}{3} \right) X_G = \frac{\pi \cdot R^2 \cdot H \cdot 3H}{3 \cdot 4} - \frac{\pi \cdot h^3 \cdot R^2 \cdot 3h}{3 \cdot H^2 \cdot 4}$$

$$\left(H - \frac{h^3}{H^2} \right) X_G = \frac{3H^2}{4} - \frac{3h^4}{H^2 \cdot 4}$$

$$(H^3 - h^3) X_G = \frac{3H^4}{4} - \frac{3h^4}{4}$$

$$X_G = \frac{3(H^4 - h^4)}{4(H^3 - h^3)}$$

4.8)



På grund av symmetri så sammanfaller masscentrum längs x-axeln.

$$\square \quad D$$

$$A_1 \quad A_2 \quad A = A_1 - A_2$$

$$A_1 = 2R \cdot 2R = 4R^2$$

$$A_2 = \frac{\pi \cdot R^2}{2}$$

$$(A_1 - A_2) X_G = A_1 \cdot X_1 - A_2 \cdot X_2$$

$$\left(4R^2 - \frac{\pi R^2}{2}\right) X_G = 4R^2 \cdot R - \frac{\pi R^2}{2} \cdot \frac{4R}{3\pi}$$

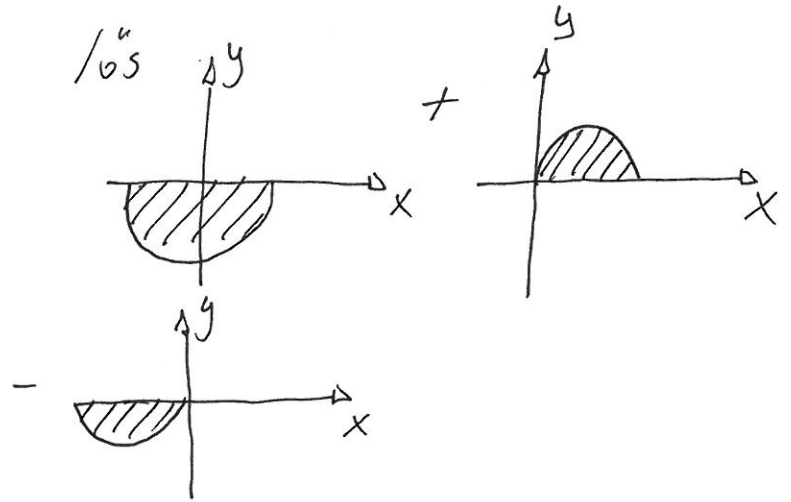
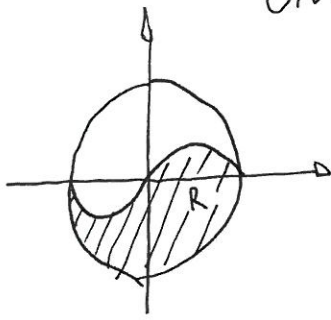
$$R^2 \left(4 - \frac{\pi}{2}\right) X_G = R^2 \left(4R - \frac{4R}{2 \cdot 3}\right)$$

$$\left(4 - \frac{\pi}{2}\right) X_G = \frac{8R}{2} - \frac{4R}{2 \cdot 3}$$

$$X_G = \frac{\cancel{24R} - \frac{4R}{2}}{\left(4 - \frac{\pi}{2}\right)} = \frac{\frac{24R}{6} - \frac{4R}{6}}{6 \left(4 - \frac{\pi}{2}\right)} = \frac{20R}{6 \left(4 - \frac{\pi}{2}\right)}$$

4,10]

Givet



$$\begin{aligned} \text{Lösning: } X_T &= \frac{\sum A_i \cdot x_i}{\sum A_i} = \frac{\frac{\pi R^2}{2} \cdot 0 + \frac{\pi \left(\frac{R}{2}\right)^2 \cdot \frac{R}{2} - \frac{\pi \left(\frac{R}{2}\right)^2 \cdot \left(-\frac{R}{2}\right)}{\frac{\pi R^2}{2} + \frac{\pi R^2}{8} - \frac{\pi R^2}{8}} \\ &= \frac{R}{4} \end{aligned}$$

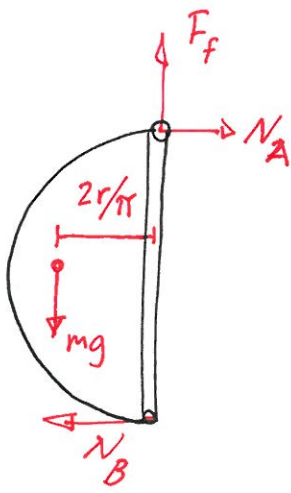
$$Y_T = \frac{\sum A_i \cdot y_i}{\sum A_i} = \frac{\frac{\pi R^2}{2} \left(-\frac{4R}{3\pi}\right) + \frac{\pi R^2}{8} \cdot \frac{4R}{3\pi} - \frac{\pi R^2}{8} \left(\frac{4R}{3\pi}\right)}{\frac{\pi R^2}{2}}$$

$$Y_T = \frac{\frac{2 \cdot \pi R^2}{2} \left(-\frac{4R}{3\pi}\right) + \frac{2 \pi R^2}{84} \cdot \frac{2R}{3\pi} + \frac{2 \pi R^2}{84} \cdot \left(\frac{2R}{3\pi}\right)}{\pi \cdot R^2}$$

$$Y_T = \frac{-\frac{4R}{3} + \frac{2R}{12} + \frac{2R}{12}}{\pi} = \frac{R \left(\frac{2}{6} - \frac{4}{3}\right)}{\pi} = \frac{R \left(\frac{1}{3} - \frac{4}{3}\right)}{\pi}$$

$$Y_T = -\frac{R}{\pi}$$

5.1)



$$\rightarrow N_A - N_B = 0 \quad (1)$$

$$\uparrow F_f - mg = 0 \quad (2)$$

$$\curvearrowright B : mg \cdot \frac{2r}{\pi} - N_A \cdot 2r = 0 \quad (3)$$

$$F_f = mg$$

$$N_A = \frac{mg \cdot 2r}{\pi \cdot 2r} = \frac{mg}{\pi}$$

Frikationsvillkoret

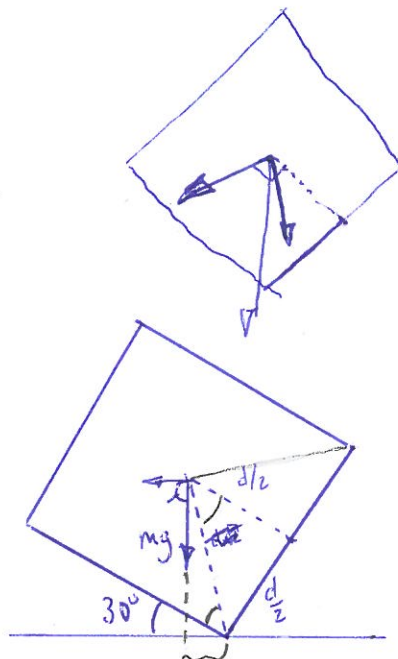
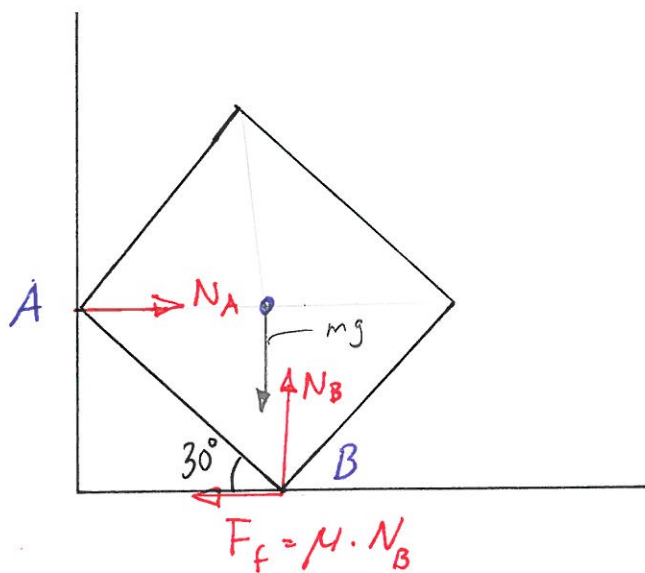
$$F_f \leq \mu \cdot N_A$$

$$F_f \leq \frac{\mu mg}{\pi} \Rightarrow mg \leq \frac{\mu mg}{\pi}$$

$$\underline{\underline{\mu \geq \pi}}$$

5.2]

Homogen kub



$$\rightarrow: N_A - F_F = 0 \quad (1)$$

$$\uparrow: N_B - mg = 0 \quad (2)$$

$$\curvearrow B \quad mg \cdot \underbrace{\frac{d}{\sqrt{2}} \cdot \cos 75^\circ}_{\text{moment arm till mg}} - N_A \cdot \frac{d}{2} = 0 \quad (3)$$

$$\sqrt{\left(\frac{d}{2}\right)^2 + \left(\frac{d}{2}\right)^2}$$

$$\sqrt{\frac{2d^2}{4}} = \frac{d}{\sqrt{2}}$$

$$F_F = N_A \Rightarrow$$

$$\frac{mg}{\sqrt{2}} \cdot d \cdot \cos 75^\circ - F_F \cdot \frac{d}{2} = 0$$

$$F_F = \frac{2mg \cdot d \cdot \cos 75^\circ}{d \cdot \sqrt{2}} = \frac{2mg \cos 75^\circ}{\sqrt{2}}$$

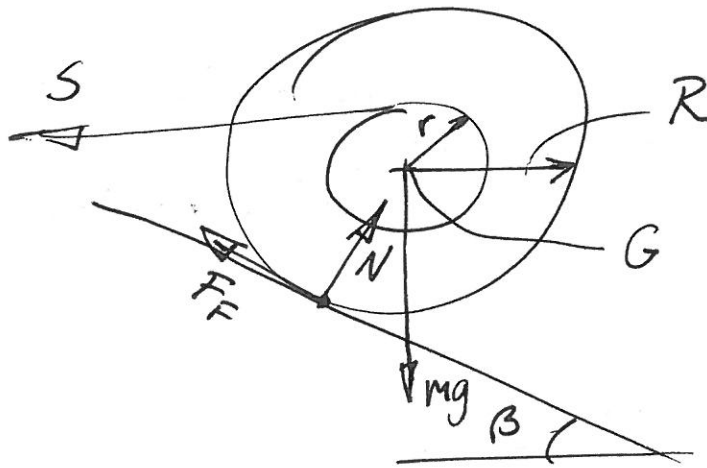
$$N_B = mg$$

$$\text{Friktionsvillkoret } F_F = \mu \cdot N_B \Rightarrow$$

$$\frac{2mg \cos 75^\circ}{\sqrt{2}} = \mu mg \Rightarrow \mu = \sqrt{2} \cos 75^\circ$$

$$\mu \approx 0,36 = \text{Ok.}$$

5,3



massan m
 y Her radie R
 inner radie r

$$S = ?$$

μ_s

$$F_{\max} = \mu_s \cdot N$$

$$\rightarrow : -F_F \cdot \cos \beta + N \cdot \sin \beta - S = 0 \quad (1)$$

$$\uparrow : N \cdot \cos \beta + F_F \cdot \sin \beta - mg = 0 \quad (2)$$

$$\odot : F_F \cdot (R) - S \cdot r = 0 \quad (3)$$

$$3 \Rightarrow S = \frac{R \cdot F_F}{r} \quad \text{ins i 1.}$$

$$\Rightarrow -F_F \cos \beta + N \cdot \sin \beta - \frac{R F_F}{r} = 0$$

$$N = \frac{(R + r \cos \beta) F_F}{r \sin \beta}$$

ins i ekv (2) ger

$$\frac{\cos \beta}{\sin \beta} \left(\frac{R}{r} + \cos \beta \right) F_F + F_F \cdot \sin \beta = mg$$

$$F_F = \frac{mgr \sin \beta}{r + R \cos \beta}$$

$$N = \frac{R + r \cos \beta}{r + R \cos \beta} mg \Rightarrow S = \frac{mgR \sin \beta}{r + R \cos \beta}$$

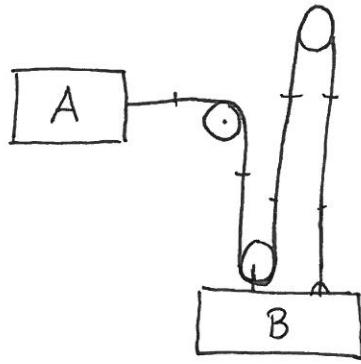


5,3 forts.) $F_F \leq \mu \cdot N$

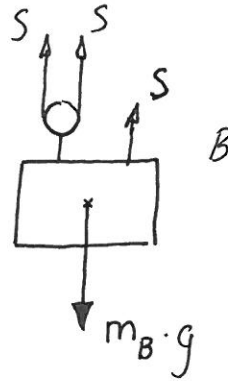
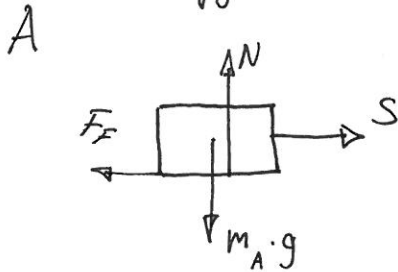
$$\frac{mgr \sin \beta}{r + R \cos \beta} \leq \mu \frac{R + r \cos \beta}{r + R \cos \beta} mg$$

$$\Rightarrow \mu \geq \frac{r \cdot \sin \beta}{R + r \cos \beta}$$

5,4]



Frilägg A o B



Jämvikt B:

$$\uparrow: 3S - m_B \cdot g = 0$$

$$S = \frac{m_B \cdot g}{3}$$

Jämvikt för A:

$$\rightarrow S - F_F = 0 \quad *$$

$$\uparrow: N - m_A \cdot g = 0$$

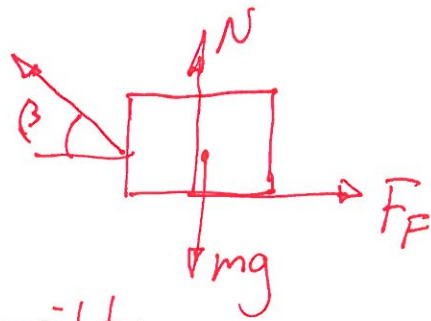
$$\Rightarrow F_F = \frac{m_A \cdot g}{3} \quad \text{och} \quad N = m_A \cdot g$$

Friktions villkoret $F_F \leq \mu \cdot N$

Vid gränsfall för glidning:

$$* \quad \frac{m_B \cdot g}{3} \leq \mu \cdot m_A \cdot g \Rightarrow \mu \geq \frac{m_B}{3m_A}$$

5.5]



jämvikt:

$$\rightarrow : -S \cdot \cos \beta + F_F = 0 \quad (1)$$

$$\uparrow : S \cdot \sin \beta + N - mg = 0 \quad (2)$$

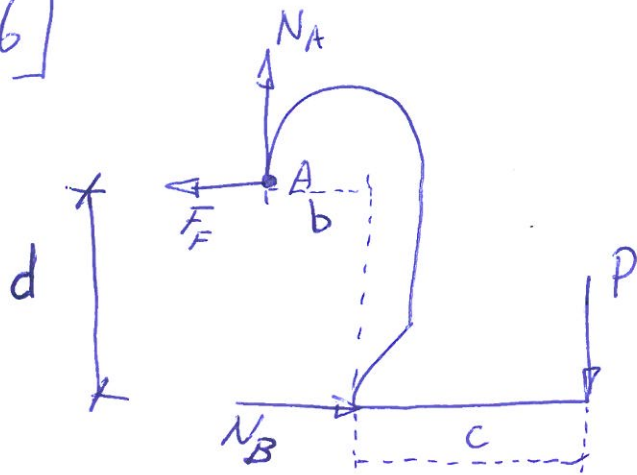
$$F_F = \mu \cdot N \quad (3)$$

$$\left. \begin{aligned} -S \cos \beta + \mu \cdot N &= 0 \\ \mu \cdot S \sin \beta + \mu N - \mu mg &= 0 \end{aligned} \right\}$$

$$\Rightarrow \mu \cdot S \sin \beta + S \cdot \cos \beta - \mu mg = 0$$

$$S = \frac{\mu mg}{\mu \cdot \sin \beta + \cos \beta}$$

5.6]



$$\rightarrow: N_B - F_F = 0 \quad (1)$$

$$\uparrow: N_A - P = 0 \quad (2)$$

$$\curvearrow A: N_B \cdot d - P(b+c) = 0 \quad (3)$$

$$N_A = P$$

$$N_B = F_F \quad \text{ins i (3)} \Rightarrow$$

$$F_F \cdot d - P \cdot (b+c) = 0$$

$$F_F = \frac{P(b+c)}{d}$$

$$F_F = \mu \cdot N_A$$

$$\frac{P(b+c)}{d} = \mu \cdot P$$

$$\mu = \frac{b+c}{d}$$

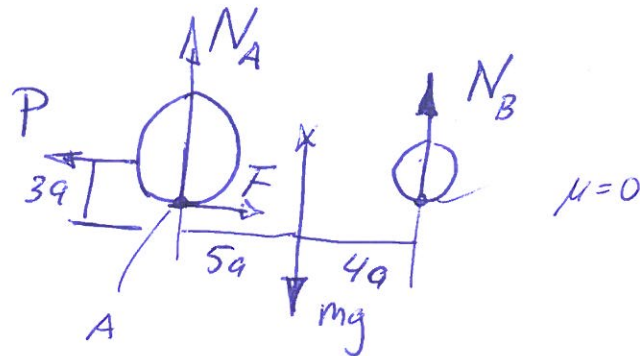
$P = \text{obegränsad}$
tills materialet går sönder.

5,8]

$$mg = 40 \text{ kN}$$

$$P = 4 \text{ kN}$$

$$\mu = 0,6$$



$$\uparrow N_A + N_B - mg = 0$$

$$\rightarrow F - P = 0$$

$$\curvearrowright mg \cdot 5a - N_B \cdot 9a - P \cdot 3a = 0$$

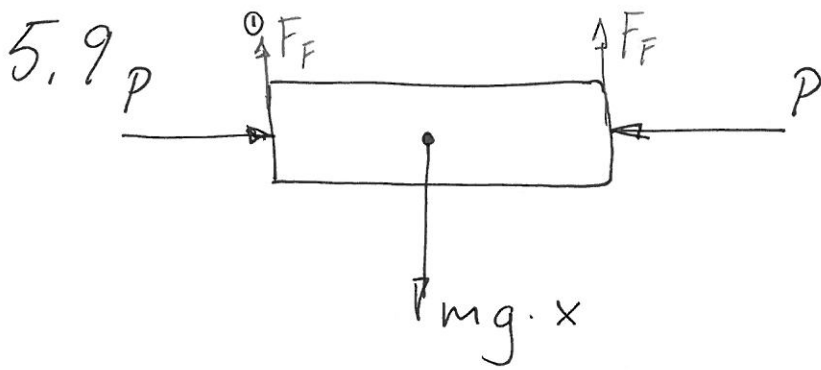
$$N_B = \frac{mg \cdot 5a + P \cdot 3a}{9a} = \frac{mg \cdot 5 + 3P}{9}$$

$$N_B = \frac{5 \cdot 40 + 3 \cdot 4}{9} = \underline{\underline{20,9 \text{ kN}}}$$

$$N_A = mg - N_B = 40 - 20,9 = \underline{\underline{19,1 \text{ kN}}}$$

$$F_F = \mu \cdot N_A = 0,6 \cdot 19,1 = 11,46 \text{ kN}$$

dvs friktionskraften är så pass stor att
 traktorn inte kommer att slira och kan
 således dra 4 kN utan problem.



$$m = 1 \text{ kg}$$

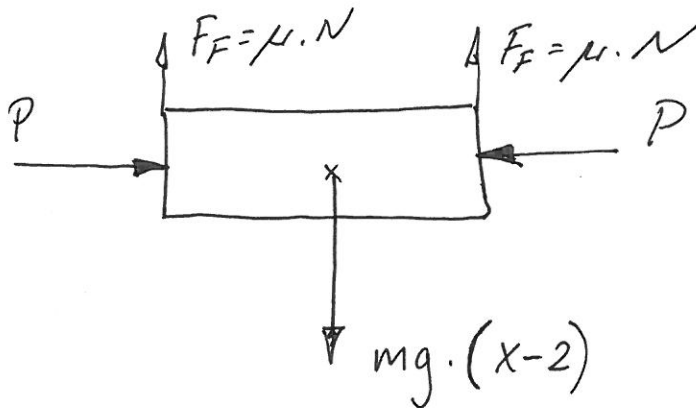
Fall 1
hend bok

$$mg \cdot x = 2 \cdot 0,5 \cdot 110$$

$$x = 11,2 = 11 \text{ böcker}$$

Fall 2

~~1.112~~ Friktion bok \leftrightarrow bok

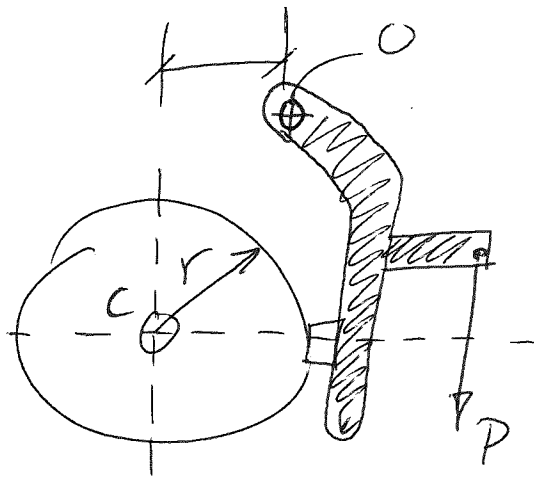


$$mg \cdot (x - 2) = 0,4 \cdot 2 \cdot 110$$

$$x = \frac{0,4 \cdot 2 \cdot 110 + 2mg}{mg}$$

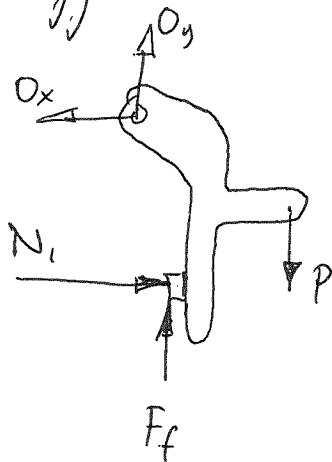
$$x = 10,96 \Rightarrow \underline{\underline{10 \text{ st böcker}}}$$

5.11)



Bestäm P om
friktkoefficienten är μ

Frilägg armen OA



$$\begin{aligned}\sum \vec{F} &= \vec{0} \\ \sum \vec{M} &= \vec{0}\end{aligned}$$

$$\curvearrowleft O: F_f \cdot (r-a) + N_1 \cdot (b) - P \cdot (c-a) = 0$$

$$F_f = \mu \cdot N_1 \Rightarrow N_1 = \frac{F_f}{\mu}$$

$$F_f (r-a) - \frac{F_f \cdot b}{\mu} - P(c-a) = 0$$

$$P = \left(\frac{(r-a) - \frac{b}{\mu}}{(c-a)} \right) F_f = \left(\frac{\mu(r-a) - b}{\mu(c-a)} \right) F_f$$

Eftersom man inte vet om hjulet roterar medurs eller moturs måste man föra in \pm i ekvationen

P blir då

$$P = \frac{b \pm \mu \cdot (r-a)}{\mu(c-a)} F_f \quad (\text{Plus då hjulet roterar moturs})$$